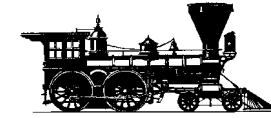


Derailment, simulation and measurement

Kik, Menssen, Moelle (ArgeCare)

Bergander (DB-AG)

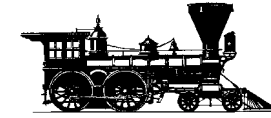
Vincent (SNCF)



Project:
**Assessment of vehicle-track
interaction
with special reference to
dynamic safety in
operating conditions
(DYSAF)**

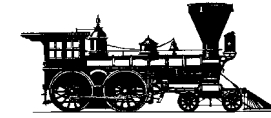
**funded by the European Commission Industrial
and Materials Technologies programme
(Brite-EuRam III)**

DYSAF – Structure



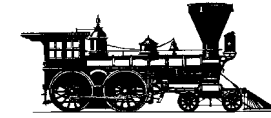
- ✂ **Design and use of a Test Running Gear for derailment tests at higher speeds**
- ✂ **Investigation of safety criteria by means of tests and MBS tools**
- ✂ **Feasibility and investigation in an Integrated Dynamic Measuring System**

DYSAF – Partners



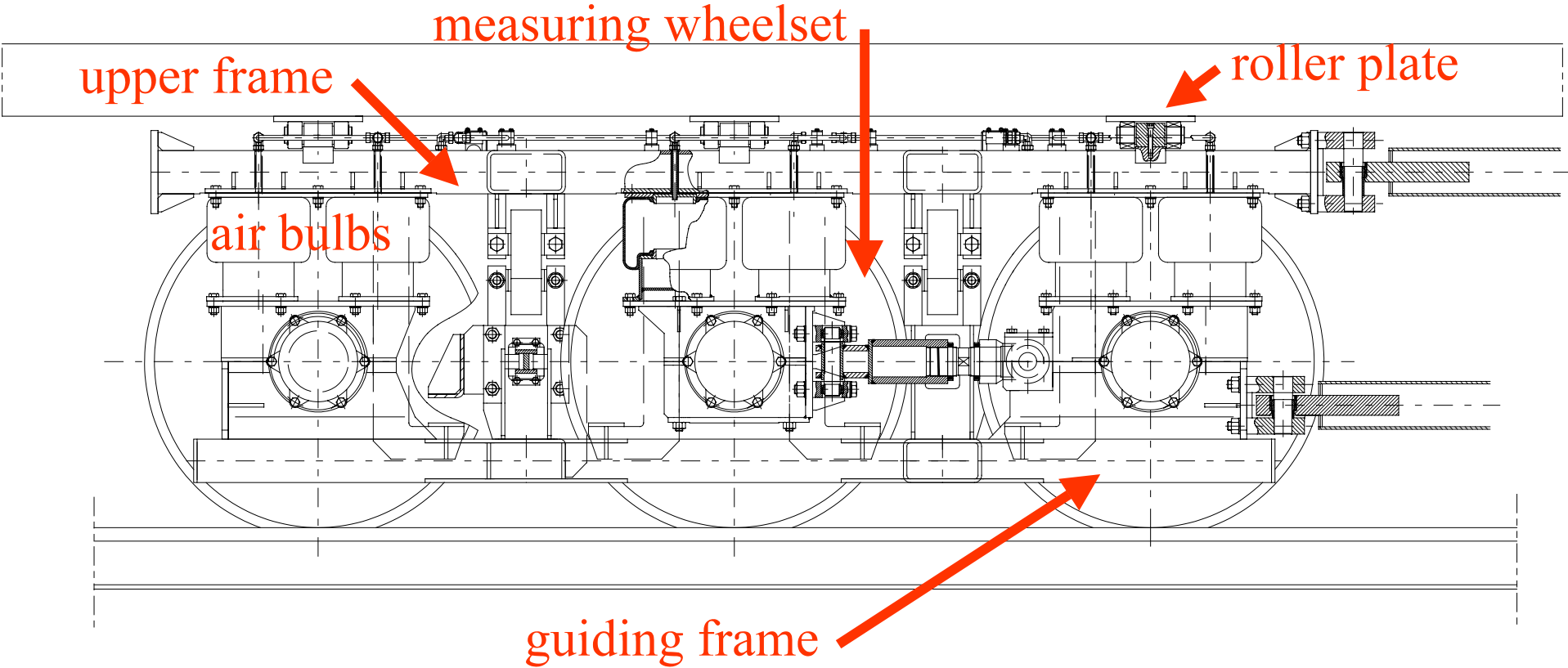
- ✂ **ERRI, ERESMAN**
- ✂ **DB AG, SNCF, ÖBB, NSTO**
- ✂ **FIAT Ferroviaria, Siemens SGP**
- ✂ **TU Graz, TU Munic, Uni. Bologna**
- ✂ **C+K, ArgeCare**

Comparison of measurements in Velim and simulation

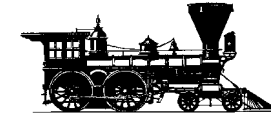


- ✂ **Test vehicle and model**
- ✂ **Test program**
- ✂ **Parameter identification**
- ✂ **Test and simulation results**
- ✂ **Conclusions**

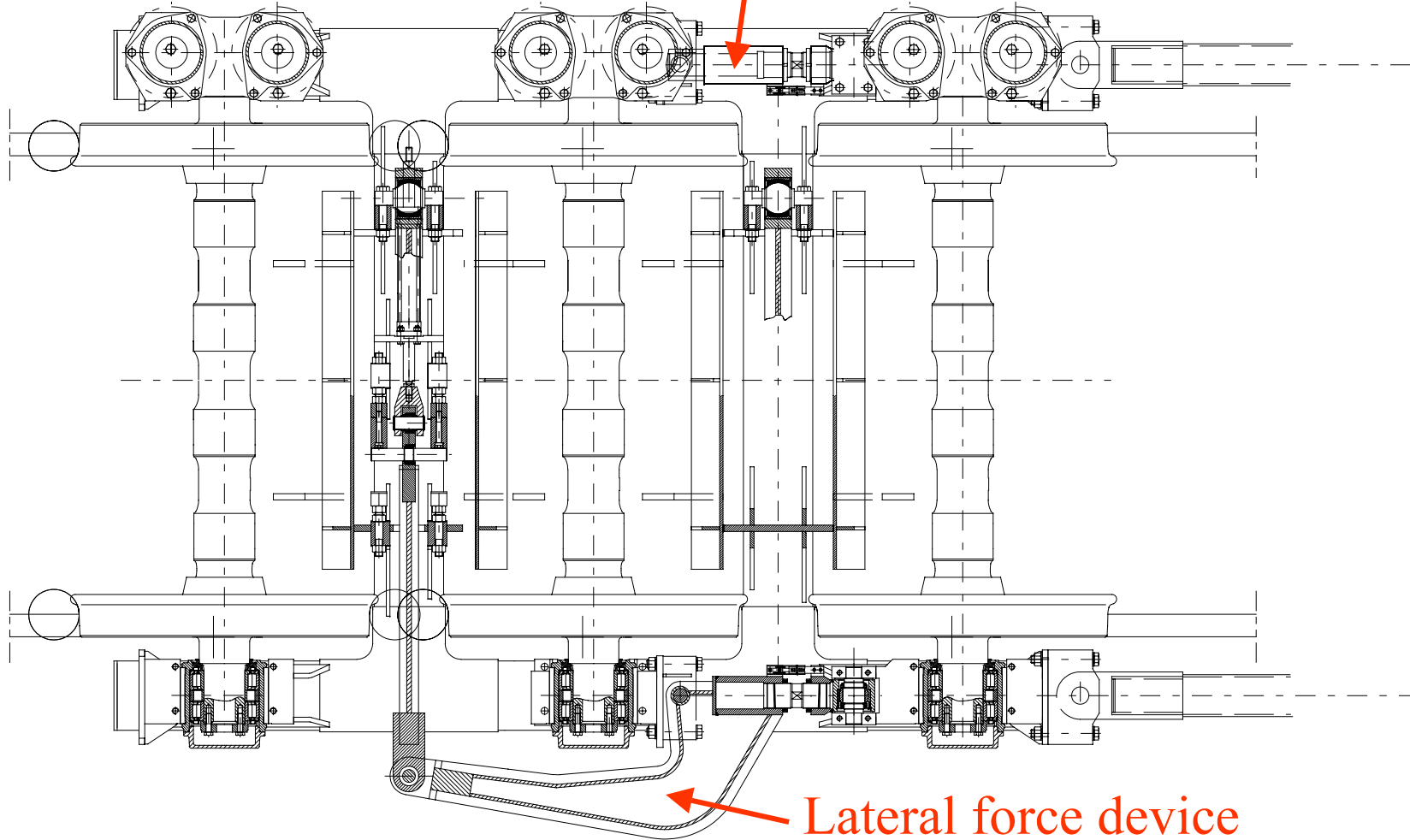
Test running gear



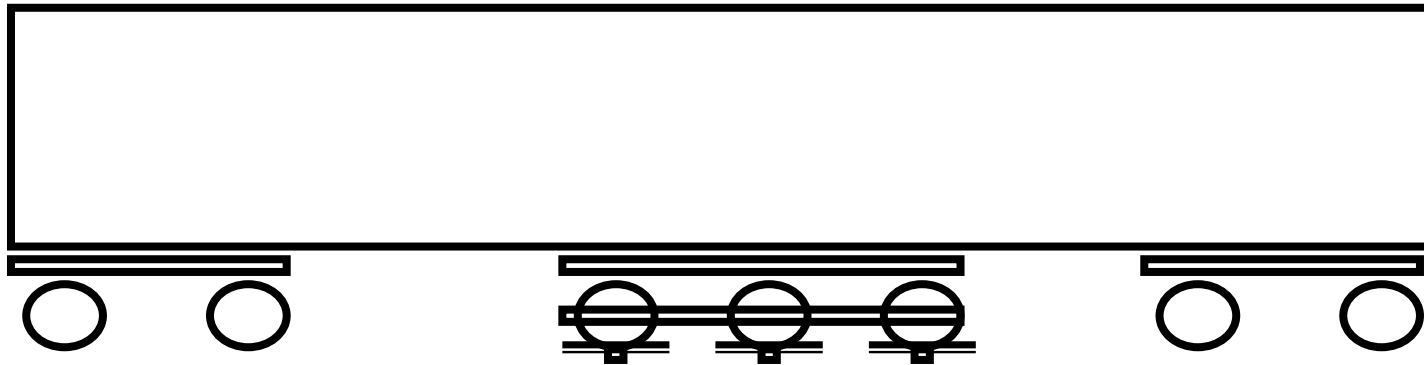
Test running gear



Adjusting screw for angle of attack

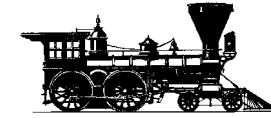


Mechanical model



- × 21 rigid bodies, 87 DOFs
- × Wheel-rail elements, non-elliptic contact patches, multi point contact
- × Different measured profile on each wheel
- × Flexible rail/sleeper model for TRG
- × Track irregularities by gauge and lateral and vertical alignment of left and right rail

Purpose of test and simulation cases

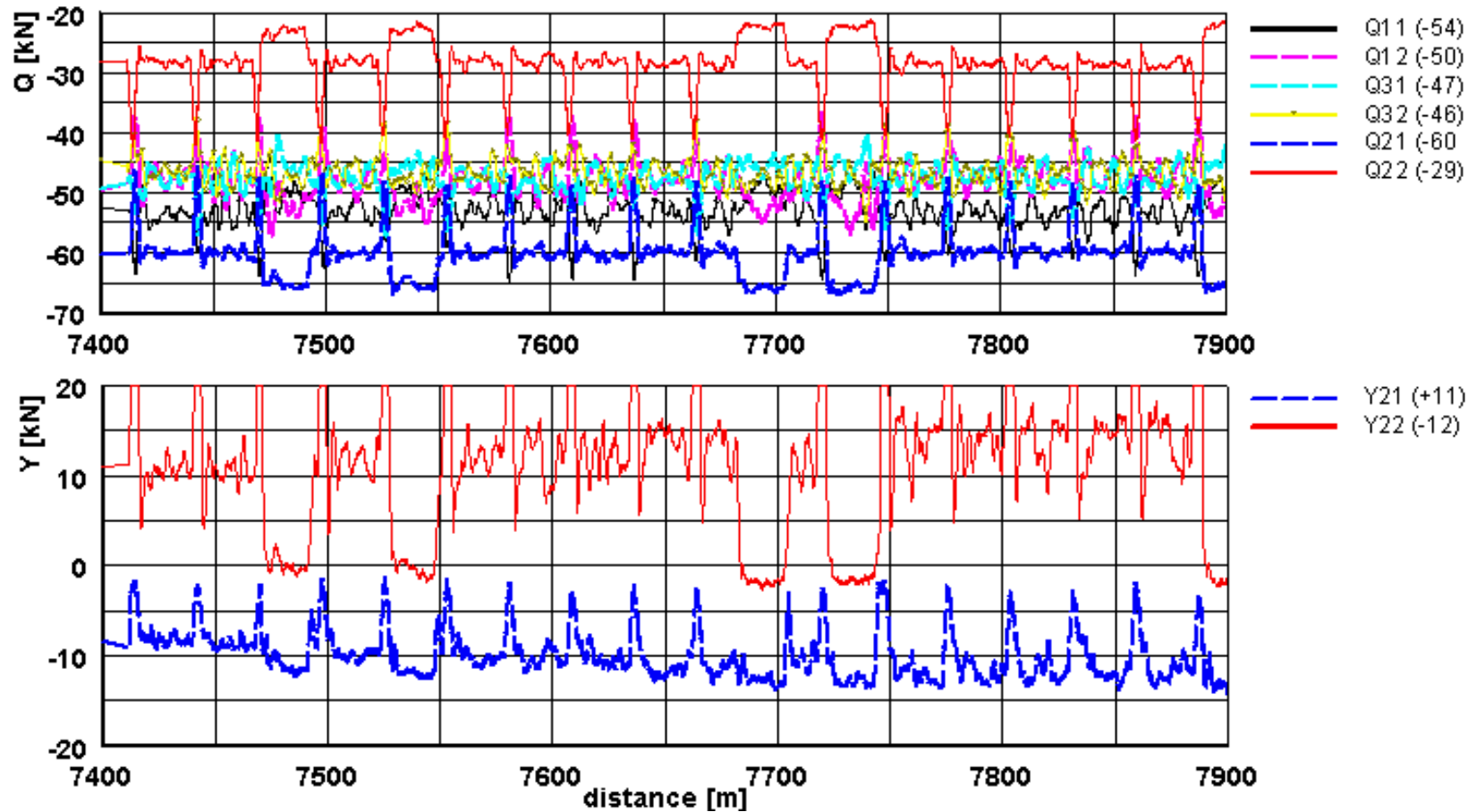


- ✂ **Investigation of safety criteria for vehicles in high speed operation**
 - ∞ **behavior of a guiding wheel,**
 - ∞ **behavior of an unloaded wheel**
- ✂ **under the effect of track irregularities**

Identification of wheel loads and friction coefficient



II2D211_001_7400_900_QY_2mMean



Wheel- rail and contact patch configuration



Track forces

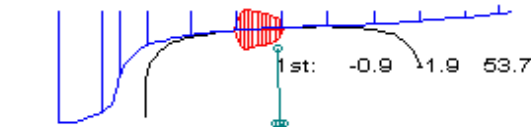
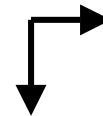
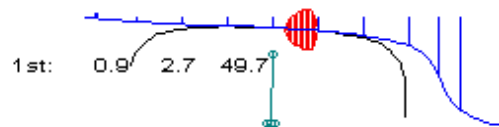
left

right

Q_{i2} Y_{i2}

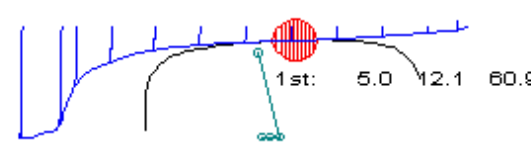
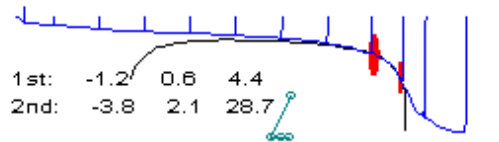
Q_{i1} Y_{i1}

50.



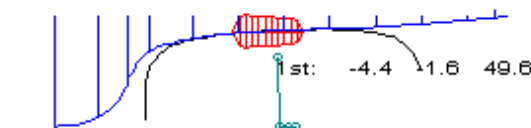
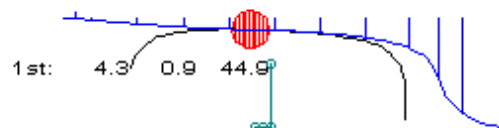
54.

29. -12



60. 12

46.



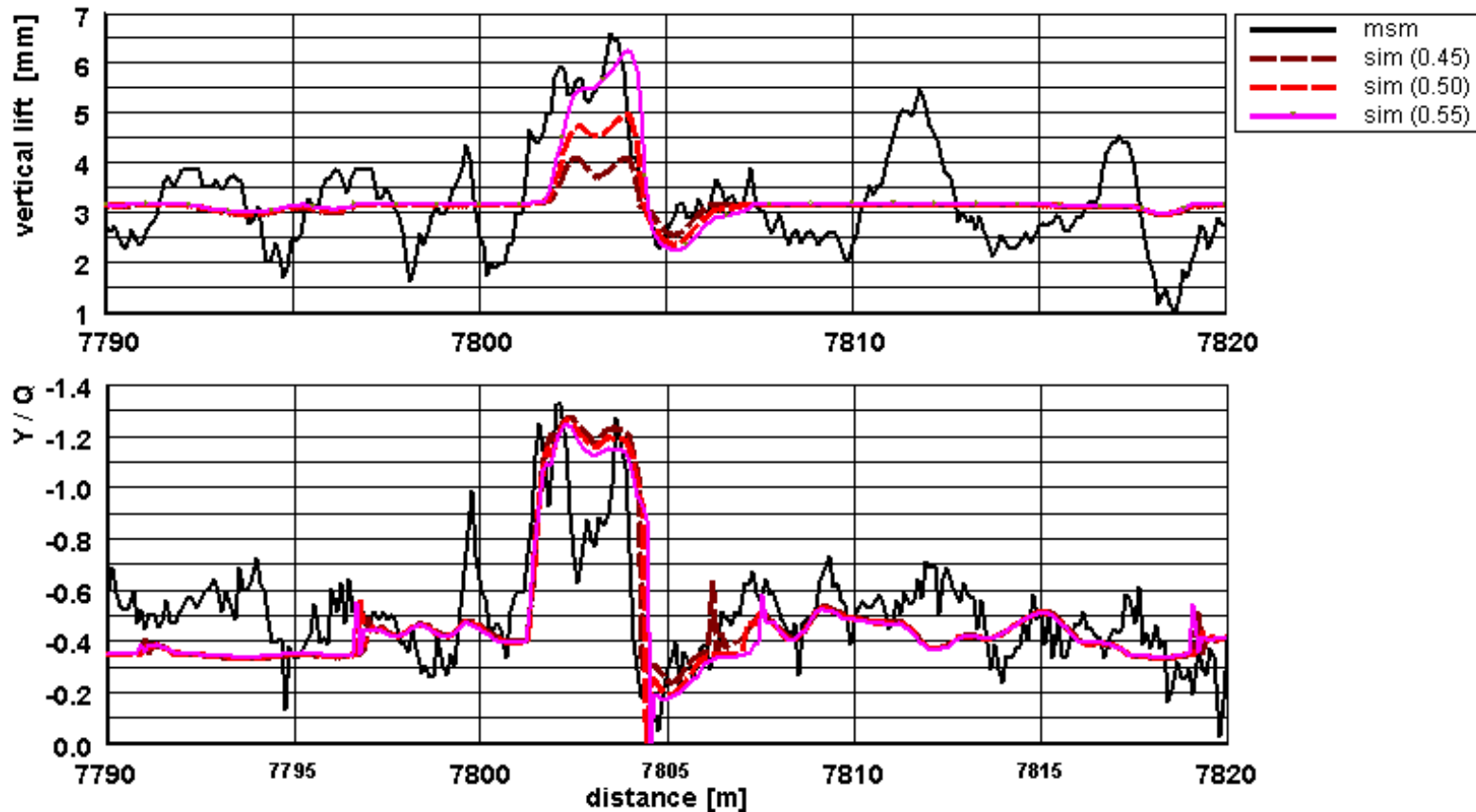
47.

Contact forces, longitudinal, lateral and normal

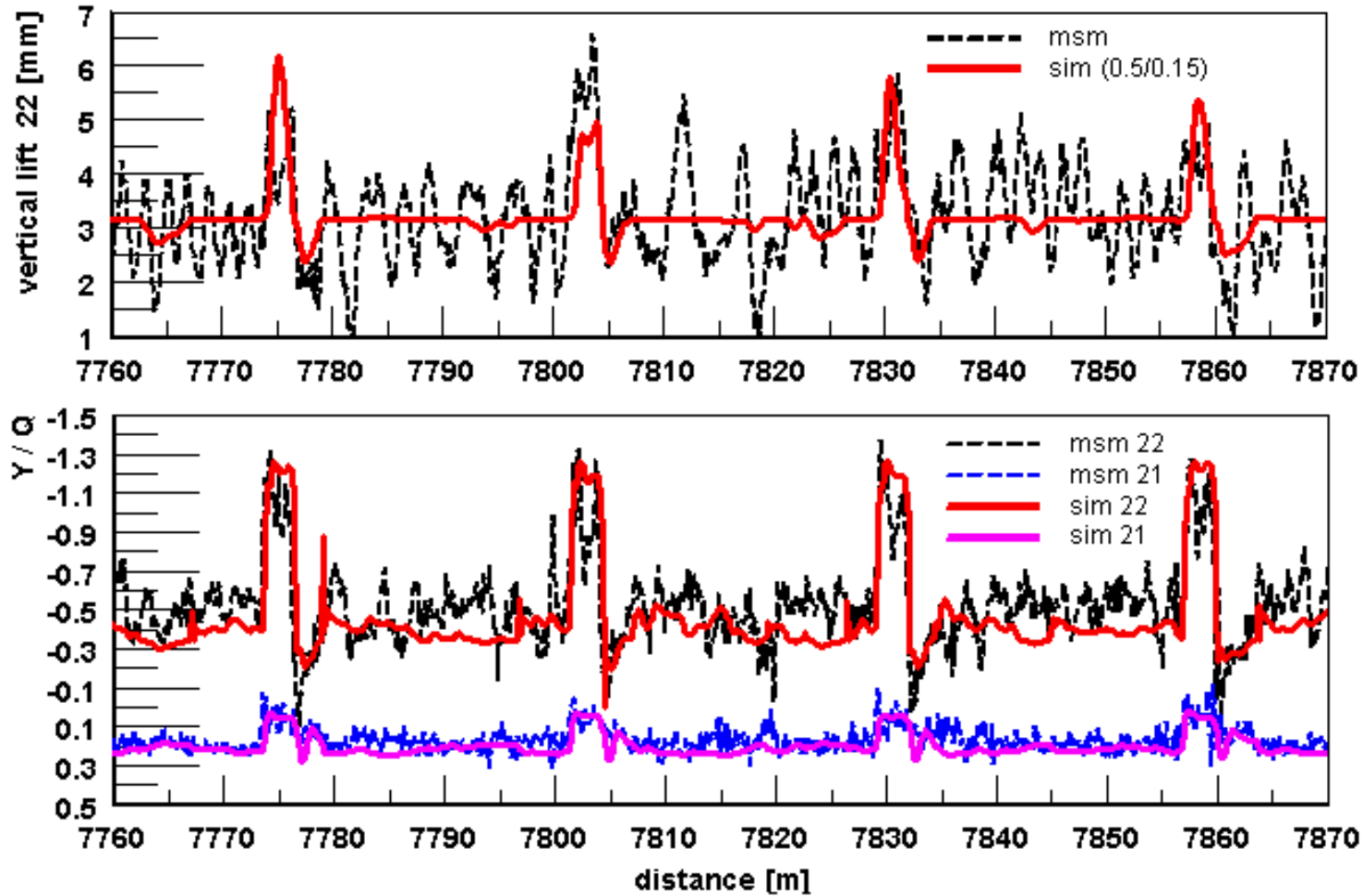
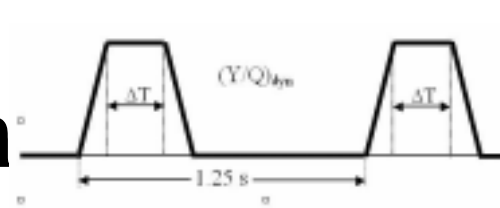
Identification of friction coefficient on flange



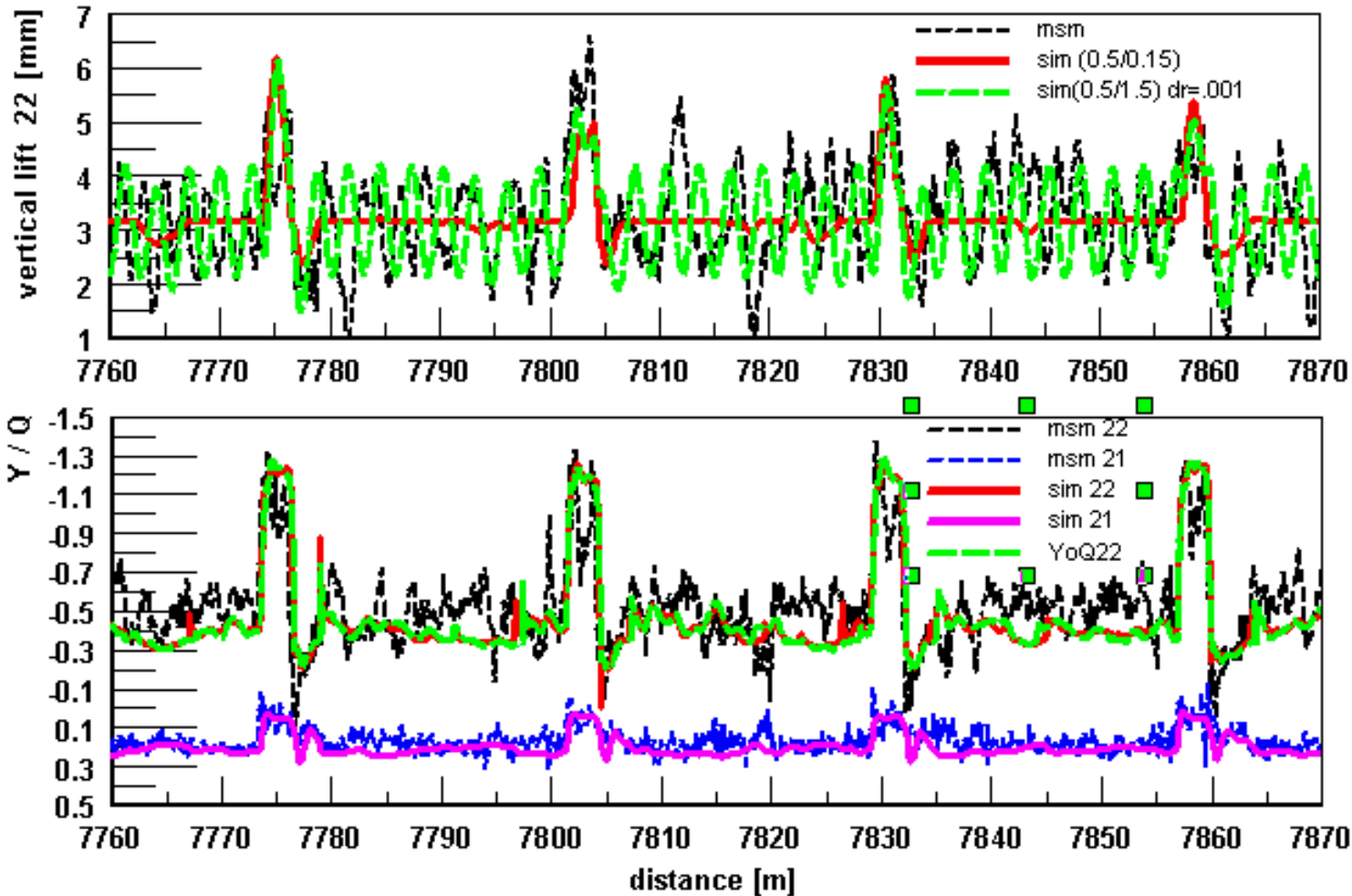
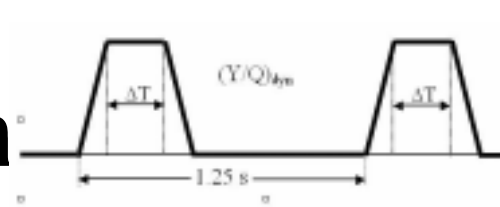
II2D211_001_one_cycle_YoQ22_dz22



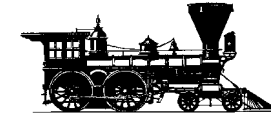
Run with 80 km/h



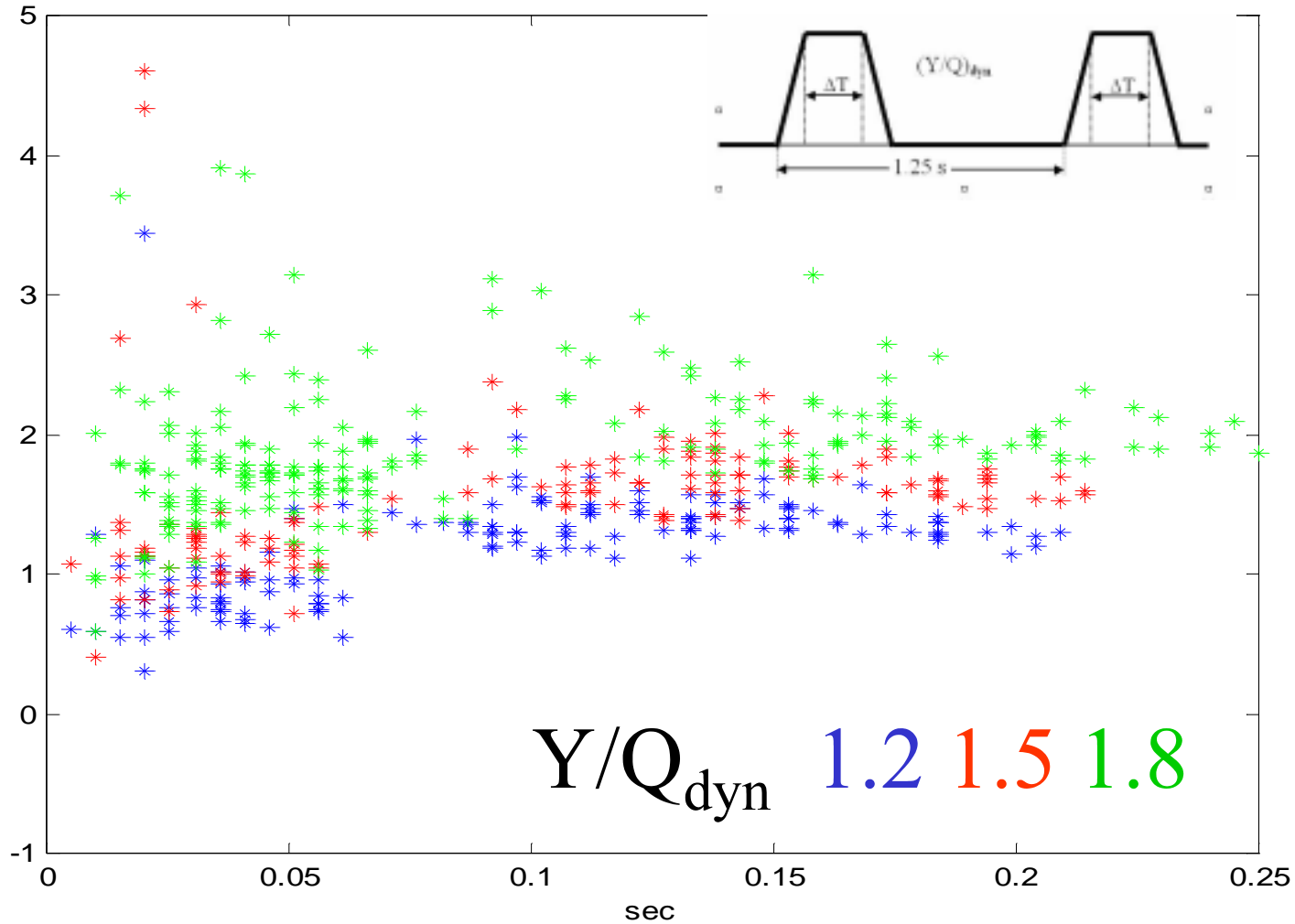
Run with 80 km/h



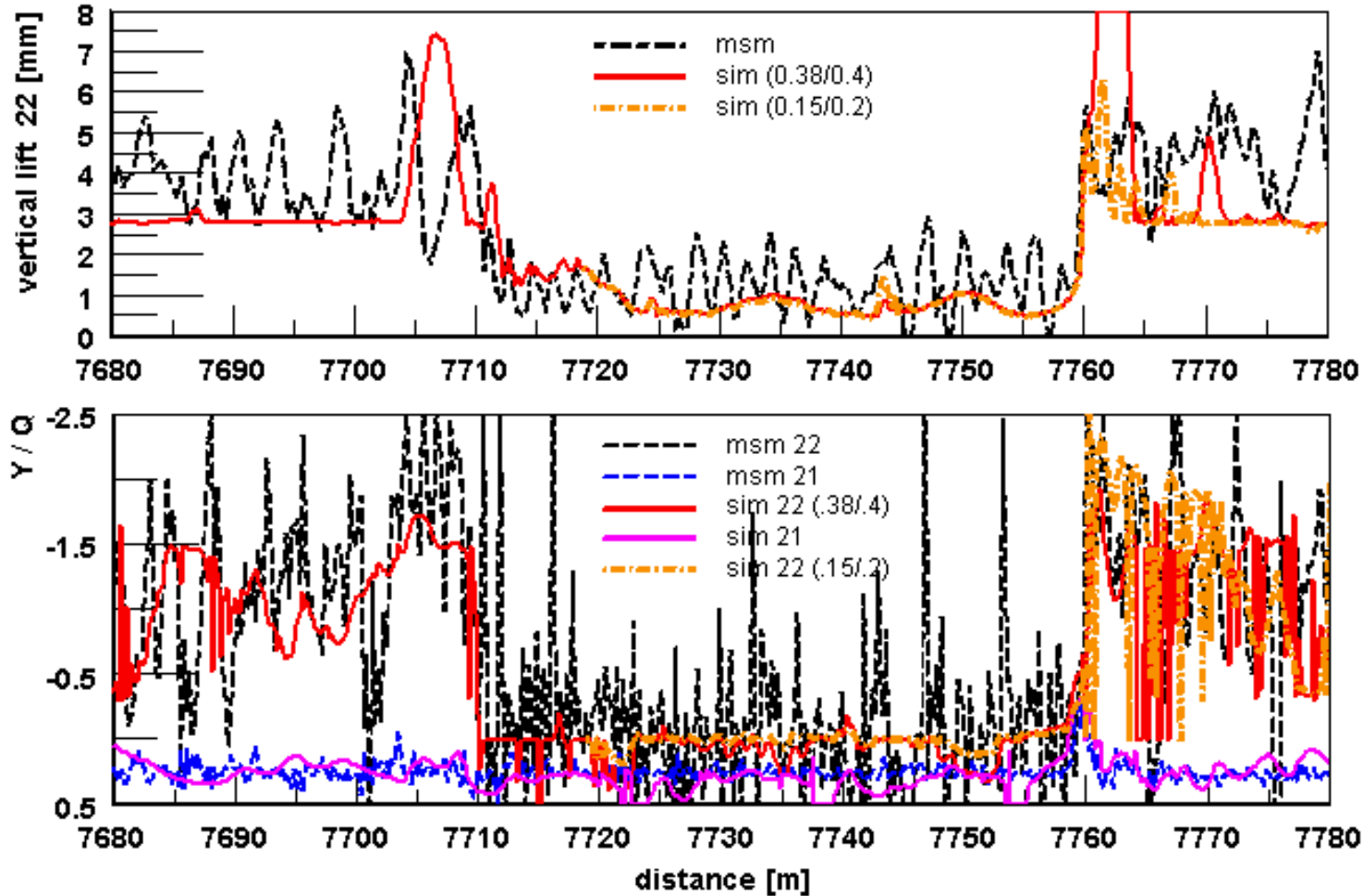
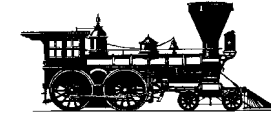
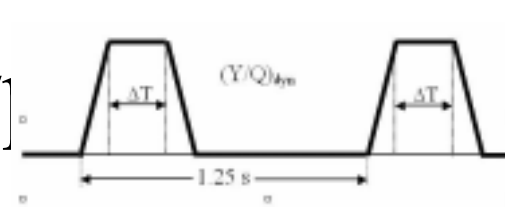
Y/Q of test runs 160 km/h



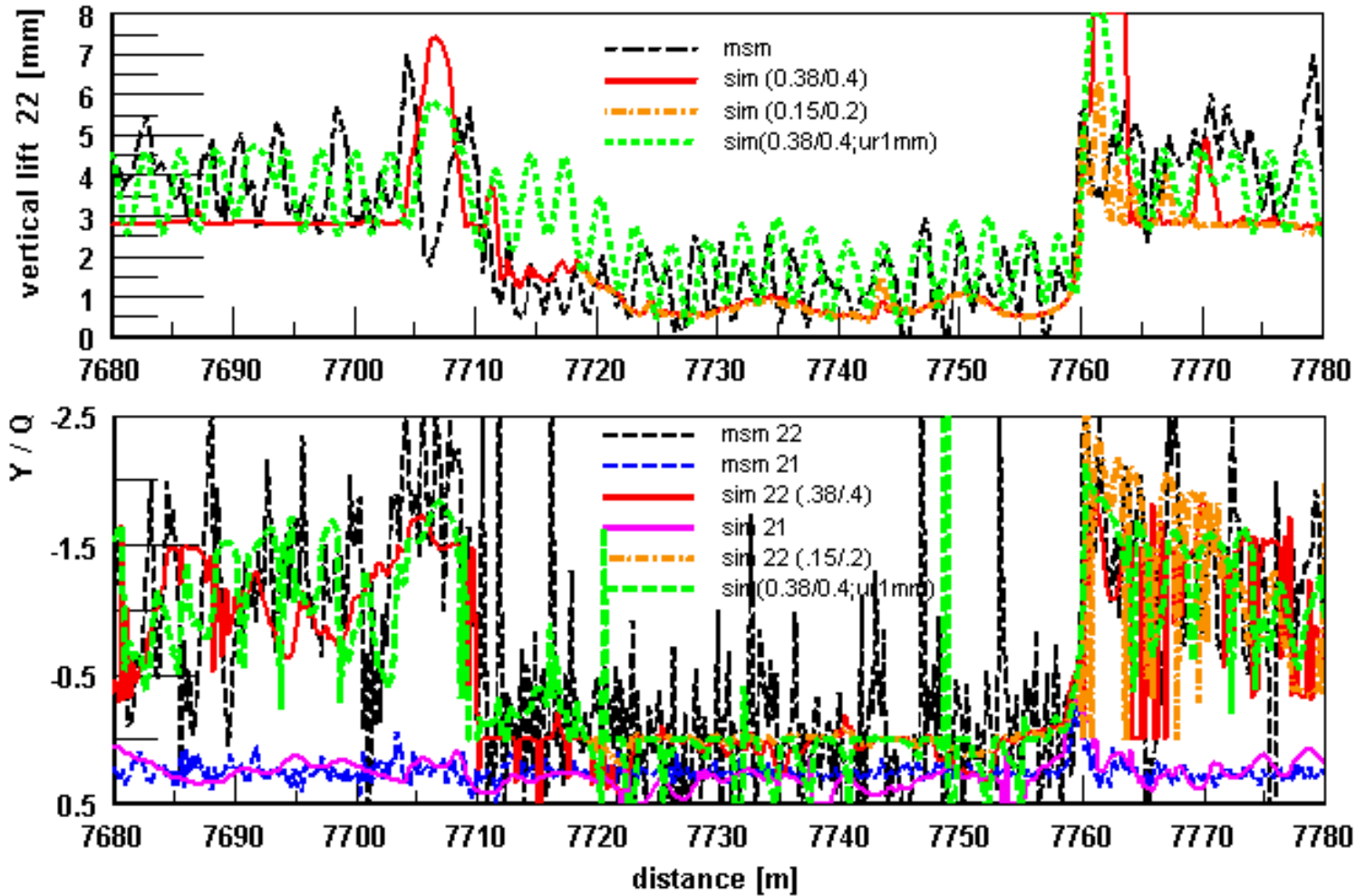
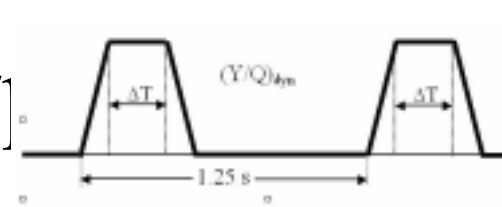
Y/Q 22 max values versus Y/Q pulse duration, speed 160 km/h



Run with 160 km/h



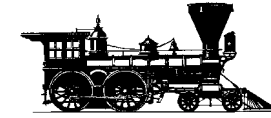
Run with 160 km/h





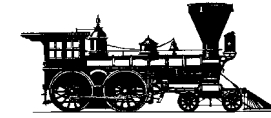
Animation of TRG, same run as
in the last diagram II2D263_001

Improvements



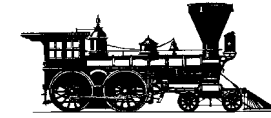
- ✘ Comparison included dynamics due to an unround wheel. But dynamics of hydraulic device should be included too..
- ✘ Measurement of track irregularities should be improved to include smaller wavelength defects, e.g using an IDMS system.
- ✘ Test data should include rotational velocity of MWS and longitudinal wheel forces to ease identification of friction coefficient for rolling contact especially when using slightly curved contact for computation of creep forces.

Conclusion



- ✘ Wheel-rail element of ArgeCare with flexible normal contact for multi point contact is close enough to reality to enable derailment simulations.
- ✘ A reasonable threshold of Y/Q as derailment criterion or a general multi parameter criteria based on Y/Q could not be derived until now.
- ✘ Simulation nowadays is a powerful tool to investigate safety of railway vehicles.

Remarks



- ✂ Thanks to EU funding and Velim test facilities it was possible to do derailment tests up to 160 km/h which did and will improve our knowledge about derailment of railway vehicles.