



Recent Advances in ADAMS/Solver



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Product Development

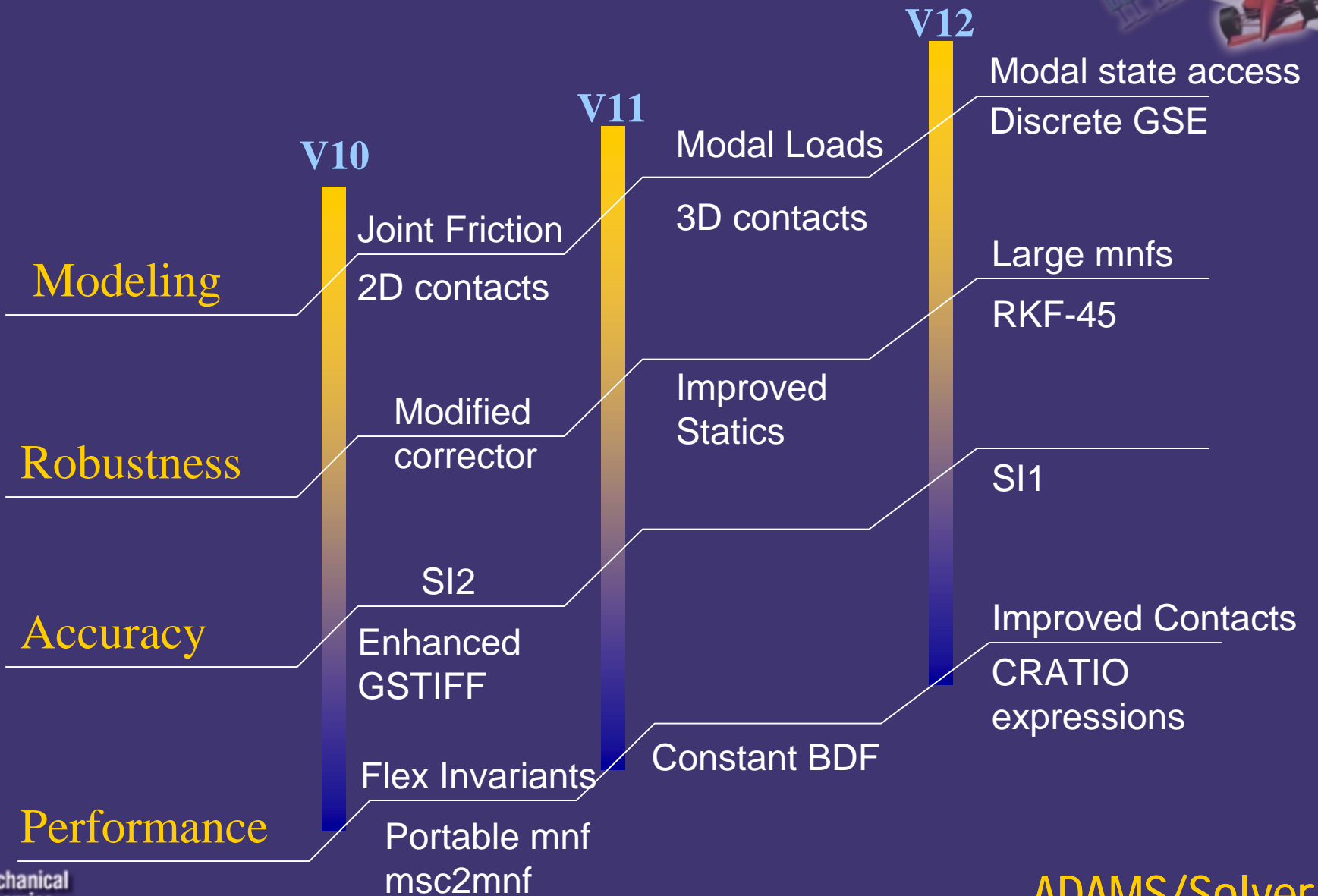
ADAMS/Solver



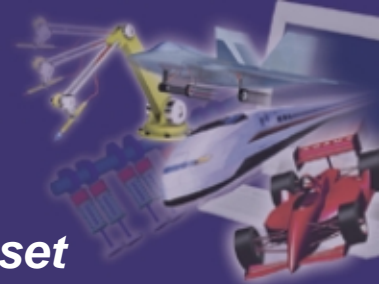
- Expanded domain for virtual prototyping
 - ◆ Controls
 - ◆ Vibration
 - ◆ Durability
 - ◆ New Applications (Driveline, Aircraft...)
- Solver Requirements
 - ◆ Expanded modeling capabilities
 - ◆ More robust solutions
 - ◆ Improved accuracy
 - ◆ Faster simulation performance



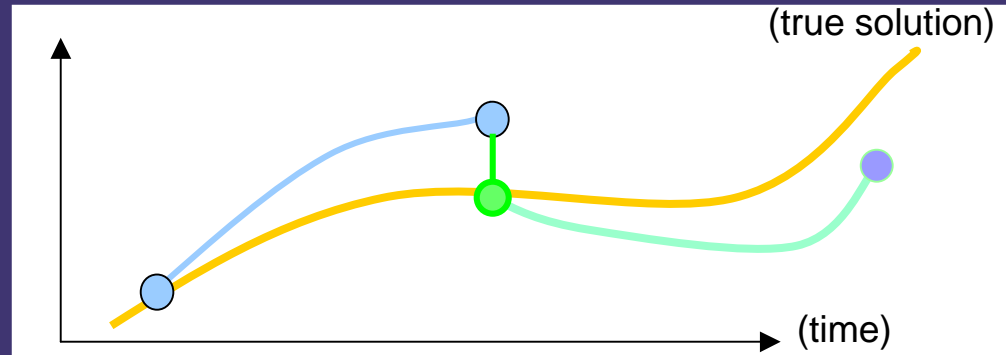
Advances in ADAMS/Solver



The ADAMS Strategy



Simpler equations preferred to fewer but more complex set



Predictor

Speed

Improved extrapolation

Corrector

Robustness

Optimize convergence criteria
Reduce index of equations

Step/Order Control

Accuracy

Reduce index of equations
Improve modeling

Numerical Methods Improvements



ADAMS v10

Gstiff

- Enhanced corrector
- Modified corrector
- SI2 formulation

ADAMS v11

Constant BDF

- Enhanced corrector
- Modified corrector
- SI2 formulation

ADAMS v12

Integrators

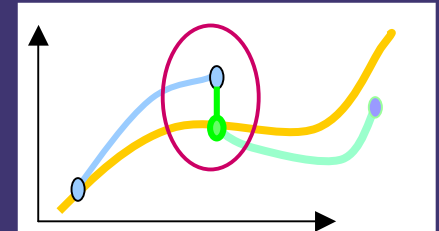
- Enhanced corrector
- Modified corrector
- SI2 formulation
- SI1 formulation
-
- RKF 4-5

Enhanced GSTIFF Integrator



■ Improved corrector

- ◆ Optimize convergence criteria
- ◆ Corrector convergence test is changed
 - Old Tolerance: $\varepsilon / 1000$
 - New Tolerance: $\varepsilon / 2 \cdot Neq \cdot (Order + 2)$
- ◆ Relative Error Calculations changed
 - Old: $Error = \Delta / MAX(1, State)$
 - New: $Error = \Delta / MAX(1, Max_State_so_far)$



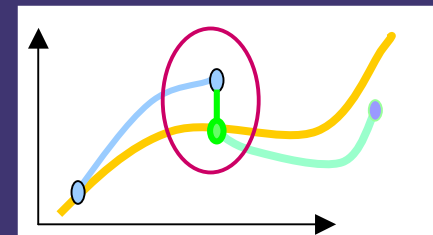
■ Real-world benefits

- ◆ Vehicle Dynamics
 - Average speed improvement of ~30%
 - No significant change in results
- ◆ Durability
 - Accurate accelerations (reduction in spikes)

Modified Corrector

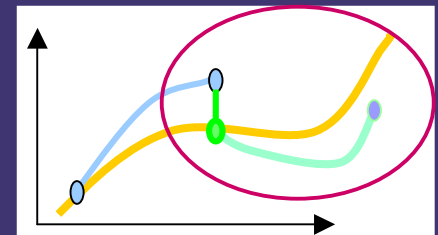


- Improved corrector
 - ◆ Optimize convergence criteria
 - ◆ Corrector monitors errors only on:
 - displacements
 - user defined dynamic states
 - ◆ Corrector does not monitor error on:
 - Velocities
 - Lagrange Multipliers
 - Forces
- Real-world benefits
 - ◆ Contact
 - 200% faster
 - Fewer corrector convergence failures
 - ◆ Friction
 - ◆ Durability





- Improved Corrector
 - ◆ Optimize state selection
- Improved Step/Order Control
 - ◆ Reduce index of DAE
- Real-world benefits
 - ◆ Nibble (wheel imbalance test)
 - 200% reduction in simulation time
 - Improved accuracy
 - ◆ Durability
 - Improved forces
 - Improved accelerations





Index 3 Formulation

$$\dot{p} - \frac{\partial L}{\partial q} + \Phi_q^T \lambda - \sum_{k=1}^{na} F_k \cdot \frac{\partial r_k}{\partial q} = 0$$

$$p - \frac{\partial L}{\partial u} = 0$$

$$u - \dot{q} = 0$$

$$\Phi(q, t) = 0$$

L = Lagrangian = T - V

Φ = Constraint Equations

r = Application point of force

p = Momenta

u = Velocity

q = Displacement

λ = Lagrange Multiplier

F = Externally applied force



SI2 Formulation

$$\dot{p} - \frac{\partial L}{\partial q} + \Phi_q^T \lambda - \sum_{k=1}^{na} F_k \cdot \frac{\partial r_k}{\partial q} = 0$$

$$p - \frac{\partial L}{\partial u} = 0$$

$$u - \dot{q} + \Phi_q^T \mu = 0$$

$$\Phi(q, t) = 0$$

$$\dot{\Phi}(q, u, t) = 0$$

L = Lagrangian = T - V

Φ = Constraint Equations

r = Application point of force

p = Momenta

u = Velocity

q = Displacement

λ = Lagrange Multiplier

F = Externally applied force

μ = Lagrange Multiplier

SI1 Formulation



$$\dot{p} - \frac{\partial L}{\partial q} + \Phi_q^T \dot{\zeta} - \sum_{k=1}^{na} F_k \cdot \frac{\partial r_k}{\partial q} = 0$$

$$p - \frac{\partial L}{\partial u} = 0$$

$$u - \dot{q} + \Phi_q^T \dot{\kappa} = 0$$

$$\Phi(q, t) = 0$$

$$\dot{\Phi}(q, u, t) = 0$$

$L = \text{Lagrangian} = T - V$

$\Phi = \text{Constraint Equations}$

$r = \text{Application point of force}$

$p = \text{Momenta}$

$u = \text{Velocity}$

$q = \text{Displacement}$

$\dot{\zeta} = \text{Lagrange Multiplier}$

$F = \text{Externally applied force}$

$\dot{\kappa} = \text{Lagrange Multiplier}$

Index 3 - SI2 - SI1 Formulations

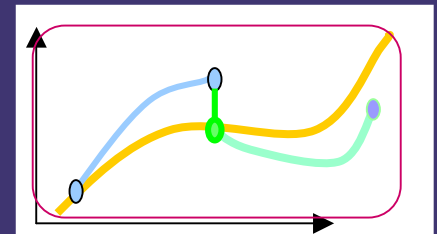


| | Index-3 | SI2 | SI1 |
|-------------------------------|--|----------------------|---|
| Accuracy | Q: High V, A: Conditional | Q, V, A: High | Q, V, A, λ : High |
| Robustness | Medium | High | High |
| Tolerance | High | Medium | Low |
| Speed | High | Medium | Medium |
| Track high frequencies | Low-Medium | High | Best |

Constant_BDF



- A new fixed-step, stiff integrator
 - ◆ User must specify HMAX
- Primarily constant step size
 - ◆ Corrector ensures solution satisfies EOM
 - ◆ Step size controls error
 - ◆ Integrator runs at KMAX(6) to minimize truncation error





Integrator/Formulation options

Release 12.0

Integrator /

$\left\{ \begin{array}{l} \mathbf{Gstiff} \\ \mathbf{Wstiff} \\ \mathbf{Constant_BDF} \end{array} \right\}, \left\{ \begin{array}{l} \mathbf{I3} \\ \mathbf{SI2} \\ \mathbf{SI1} \end{array} \right\}, \mathbf{corrector} = \left\{ \begin{array}{l} \mathbf{original} \\ \mathbf{modified} \end{array} \right\}$

RKF45

ABAM

Contact Modeling Improvements



ADAMS v10

2D Contact

- Point
- Line
- Curve
- Friction

ADAMS v11

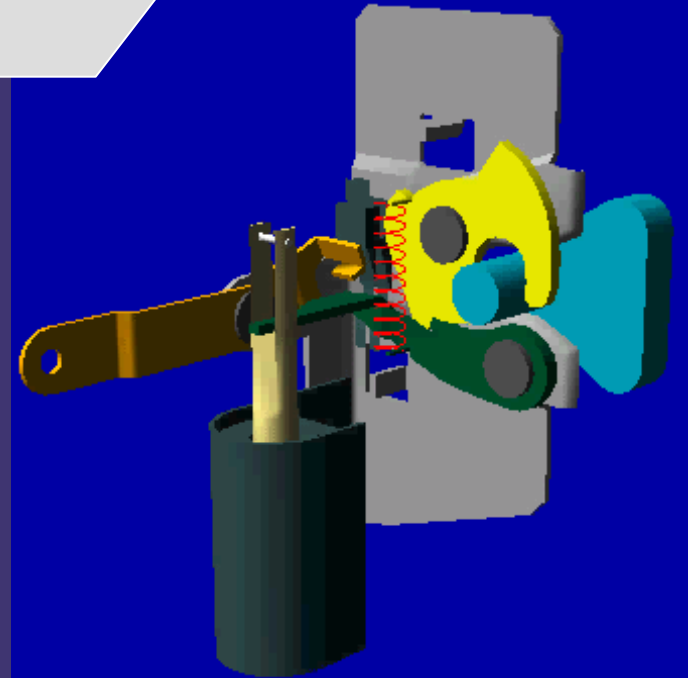
3D Contact

- Sphere
- Cylinder
- Box
- Solid

ADAMS v12

Performance

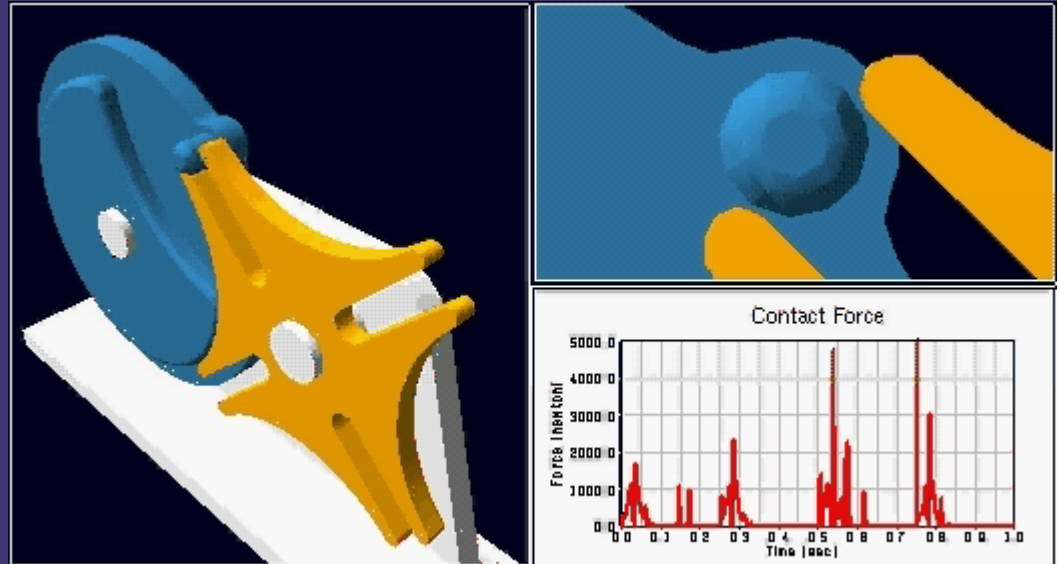
- Robustness
- Speed
- Contact Function



Contact Modeling



- Normal Force Models (Impact or Impulse)
- Simple Friction Model (No Stiction)
- Special Integrator Handling
 - ◆ Contact Sensing
 - ◆ Step-size and order control
- Extra Contact Output Steps



Flex Body Improvements



ADAMS v10

- User defined damping
- Improved NASTRAN Support
- Portable MNF's
- Invariant control

ADAMS v11

- Modal Force Loads
- Improved Abaqus Interface
- Preloaded Bodies

ADAMS v12

- Modal damping
- Large MNF (> 2GB)
- Modal State Access
- C++ Solver Support
- Mforce contour plots

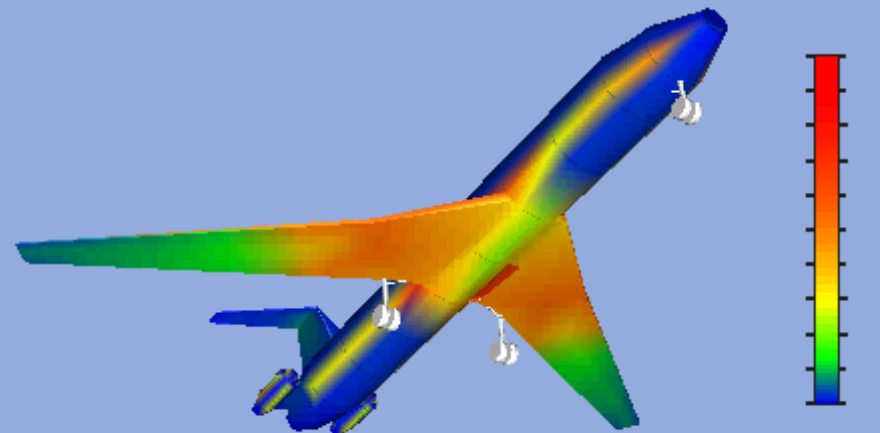
A/Flex Improvements

Release 12.0



- Control damping ratio using an expression
- Large MNF file support (> 2GB)
- Modal State access from user subroutines
- Flex Bodies in C++ Solver
- Contour plots for MFORCE in PPT

Modal Force Countours

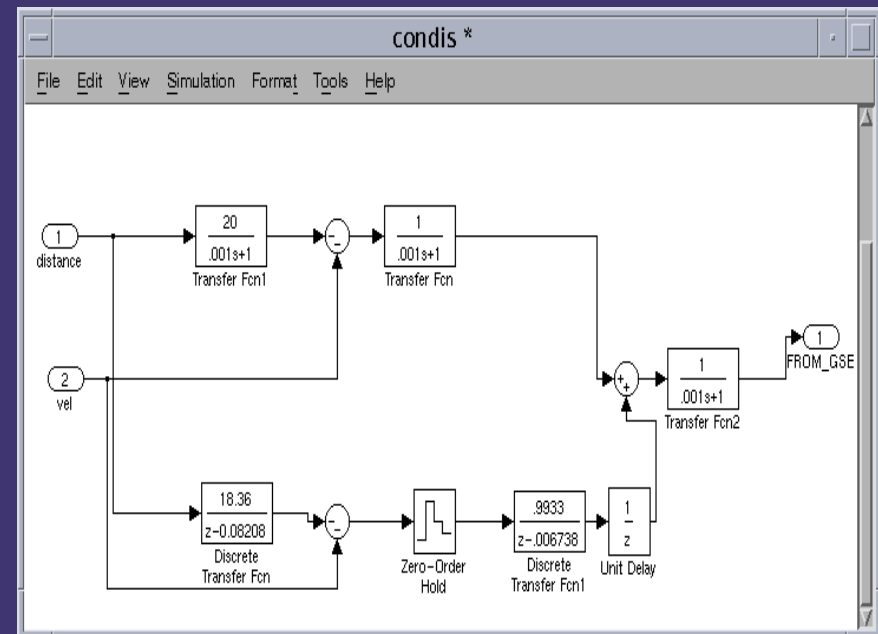


Enhanced GSE

Release 12.0



- Support for:
 - ◆ Continuous systems
 - ◆ Discrete systems
 - ◆ Combined (Sampled) systems
- New user subroutine interface
- Subroutine interface similar to Matlab/RTW
- RTW interface available with ADAMS/Controls



$$\dot{x}_c = f_c(x_c, u, t), \quad x_c(t_o) = x_{co}$$

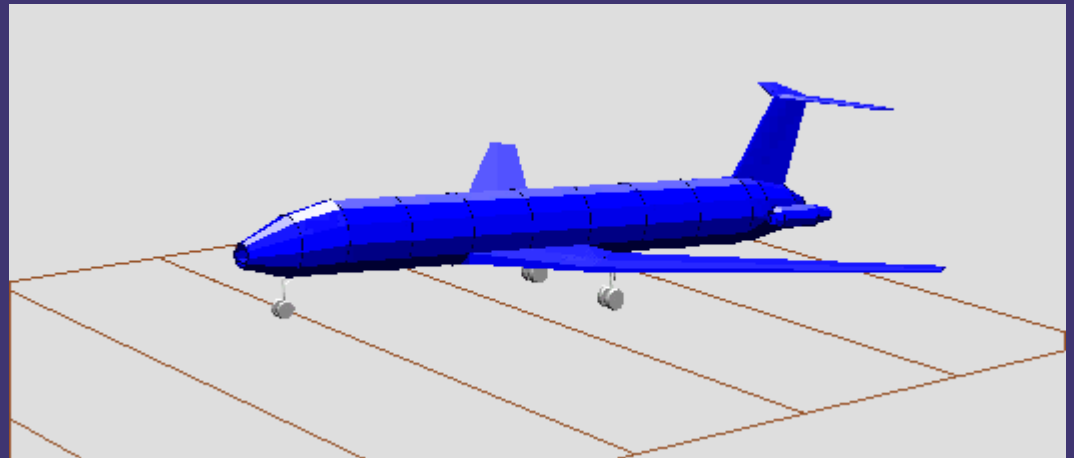
$$x_{d+1} = f_d(x_d, u, t), \quad x_d(t_o) = x_{do}$$

$$y = g(x_c, x_d, u, t)$$

ADAMS/Durability Support Release 12.0



- Enhanced FEMDATA statement
 - ◆ von Mises stress and strain
 - ◆ Loads & Nodal Deformation
 - ABAQUS
 - ANSYS
 - NASTRAN
 - ◆ Modal deformation
 - NASTRAN
 - ANSYS



The C++ Solver

Release 12.0



- Next generation ADAMS Solver
 - ◆ Object oriented structure
 - ◆ Compact and efficient
 - ◆ Facilitates encapsulation of data and methods
 - ◆ Set of modeling elements is user extensible
 - ◆ Reduced development time and maintenance costs
- Commercially available for 4 years
 - ◆ Solver for embedded products (CAD)
 - ◆ Larger user base than FullSim!
- Well-defined Interface (API)
 - ◆ Embeddable in client programs
 - ◆ CAD, Vertical products, User “main” programs

The C++ Solver

Release 12.0



- Optionally available to users since version 10
- User subroutine support (FORTRAN, C, C++)
- I/O file support
- All modeling elements except
 - ◆ Forces: NFORCE, MFORCE, FRICTION
 - ◆ Bodies: POINT_MASS
 - ◆ Integrators: ABAM, SI1, CONSTANT_BDF
 - ◆ Other: SENSOR
- Most simulation commands supported



The C++ Solver

Release 12.0



Why should I use the C++ Solver ?

By exercising the new Solver and communicating your experience you can help MDI provide you a better solver

Benefits of the C++ Solver



- Superior Modeling
 - ◆ API for Class Library - User extensibility
 - ◆ Superior implementation for Flexible Bodies
 - ◆ 2D Part
 - ◆ Generalized ACTIVATE/DEACTIVATE
 - ◆ User defined constraints
- Accuracy and Robustness
 - ◆ Analytical differentiation of expressions & user subroutines

Release 13.0



- Product migration to C++ based Solver
 - ◆ Full support for adm, acf, user subroutines
 - ◆ Default solver for selected ADAMS Vehicle Dynamics and Chassis events
- Continued improvements to the C++ Solver
 - ◆ Speed
 - ◆ Compatibility
- Improved Contact modeling, performance and post-processing



Questions?

