

Mechanical

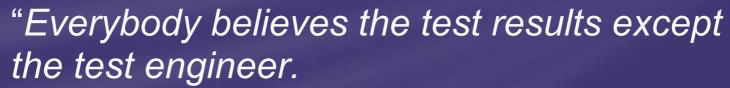


Michael Hoffmann Diego Minen



Analysis versus Testing





Nobody believes the analysis results except the analyst."

"We need to get analysis and testing on the same page"

Mike Racicot, Total Vehicle Analysis Engineer, GM





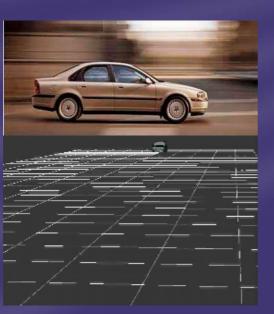
Partnership Goals

- Integrate Physical and Virtual Testing to Create the Analytical Design and Validation Process
 - Agree on and support standard interfaces
 - EDM Seamless Integration of Component or Subsystem Physical Test Results into the Functional Digital Vehicle
 - VTL Consistent Setup, Test and Analysis between Simulated and Physical Testing
 - Expertise in Integrating Solutions for Analytical Design and Validation Processes
- Bring a New Level of Productivity to Design and Development Processes

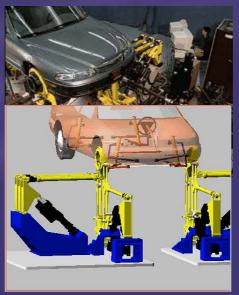




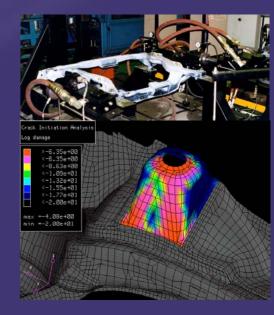




Full Vehicle Test Lab



Component **Test Lab**



- Seamless transition
 - from test track to test labs
 - from virtual to real





Software Components of a Durability Solution

BASIC COMPONENTS

ADAMS/FullSim

ADAMS/Durability

Finite Element Programm

FE-Fatigue

ADAMS/Car

ADAMS/Driveline

ADAMS/Engine

GENERIC

AUTOMOTIVE

ADAMS/Flex

ADAMS/EDM

GENERIC

ADAMS/Tire FTire

Virtual Test Lab

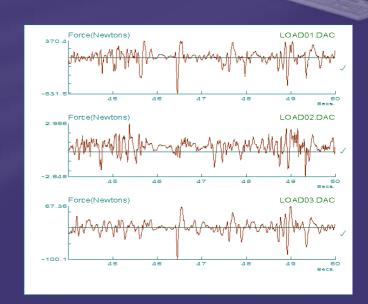
AUTOMOTIVE





ADAMS/Durability

- Reference data from DAC or RPC III files and interpolate during simulations
- Save REQUEST data to DAC or RPC III files
- Export ADAMS results to DAC or RPC III files
- View header information and/or data in DAC or RPC III files
- Plot DAC or RPC III time history data and compare with ADAMS results







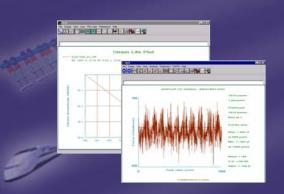
FE-Fatigue







View model and stress/strains in Pre/Post Processor

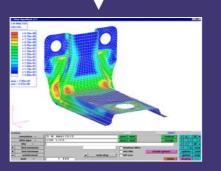






FE-Fatigue

Load and Material information



Post process fatigue results



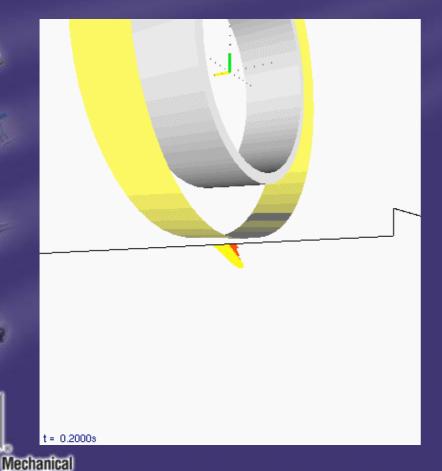


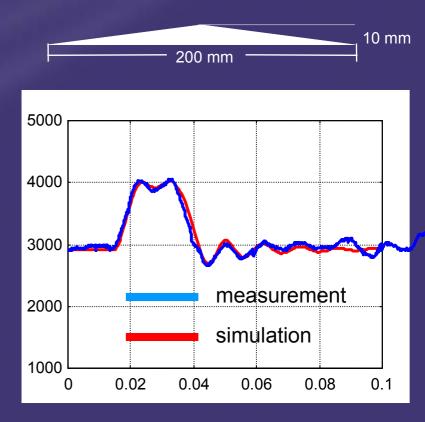
Dynamics

ADAMS/Tire FTire



Tire model for NVH and durability applications





wheel load [N]



Empirical Dynamic Model (EDM)









Tires









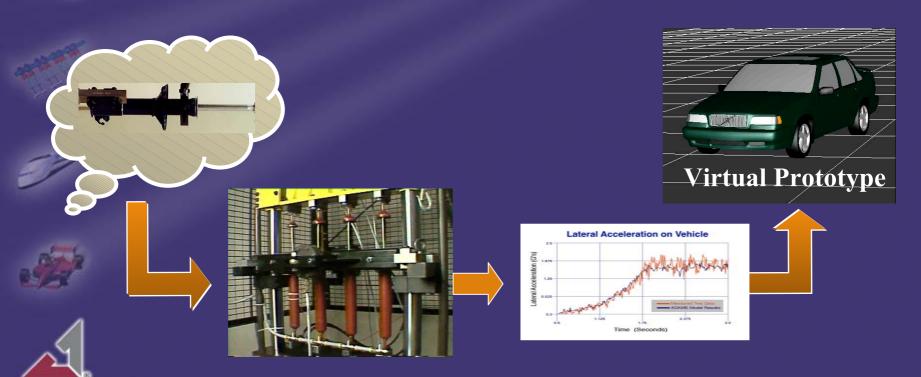




Empirical Dynamic Model



- Physical Testing Generates Data for Model Creation
 - Black Box, Characteristic, and Concept Model

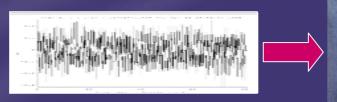




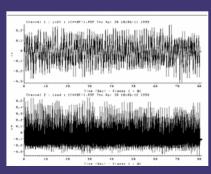
Empirical Dynamic Model



- Support of EDM
 - ◆ ADAMS/View
 - ◆ ADAMS/Car
 - ADAMS/Pre

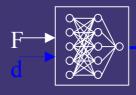


Random Displacement Command

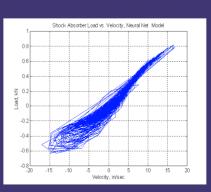


Measured Force and Displacement





Model Build

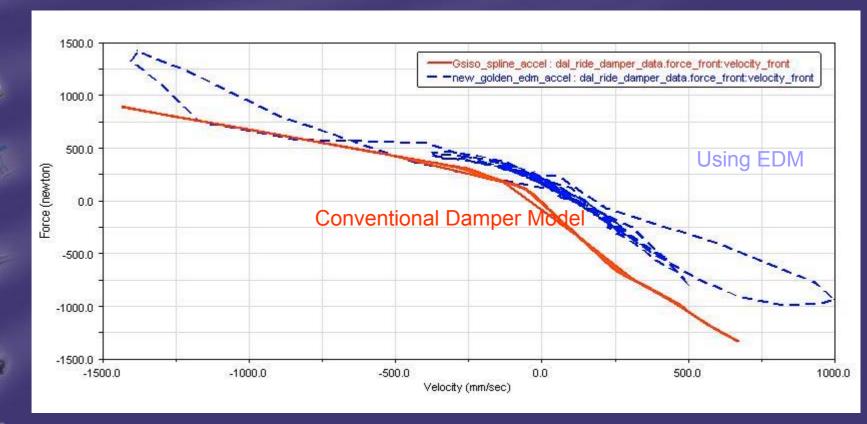


Predicted Results



Damper Loads During a Pothole Event







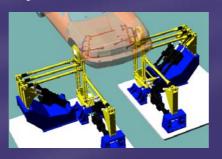


Virtual Test Lab



- Dynamic Models of MTS Systems
 - ♦ Mechanical, Hydraulic, and Control Systems









- Interface to Functional Digital Vehicle
- Common Pre and Post-processing Utilities
- Common Drive Files for Physical and Virtual Tests
- Predictive Analysis for Validation Path
- Knowledge of Test Lab engineers can be introduced much earlier in the design process





VTL Application Table



<u>Input</u>	Test Rig Model	<u>Applications</u>
Spindle Loads	No VTL Model	Fatigue Prediction
Actuator Displacements	VTL – Elasticity and Kinematics of Test Rig	Validation with Traditional Instrumentation or SWIFT
RPC Response File	VTL/RPC – Model of Controller and Hydraulics	Full RPC Iteration and Drive File Creation



How Do You Manage Your Durability Process













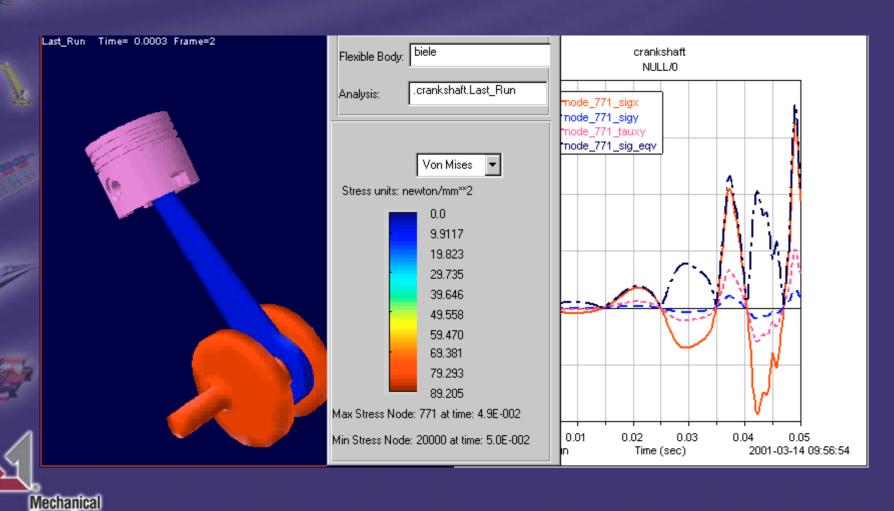


Dynamics

The Modal Stress Recovery (MSR)



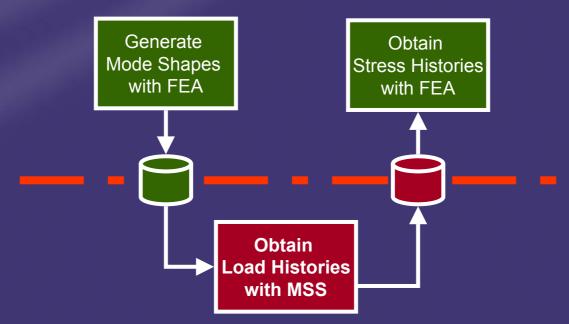
Streamlines stress and fatigue evaluation





Common Process for Stress Computation

- Parameter studies where objectives include component stresses require switching between FEA and Mechanical System Simulation
 - no integrated process
 - parameter studies and optimization on a system level is too cumbersome



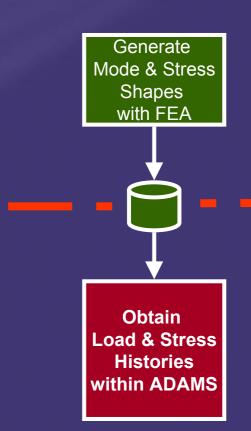




Integrated Process for Stress Computation



- The MSR Toolkit allows
 - Stress computation within the ADAMS environment
 - Parameter studies and optimization on a system level
- No constant switching between FEA and MSS
- FEA is touched only once







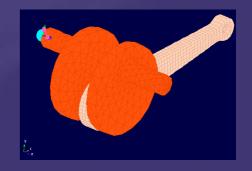
Integrated Process for Stress Computation (cont.)



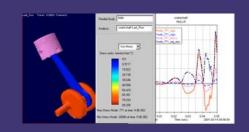
- Step One:
 - Mesh components
- Step Two:
 - Run FEA to generate mode and stress shapes
 - -> data is written to the MNF-file



- Run ADAMS to obtain the component load histories (in terms of Modal Coordinates)
- Step Four:
 - Run ADAMS/PPT to
 - compute and plot nodal stresses
 - display and animate components' stress contours







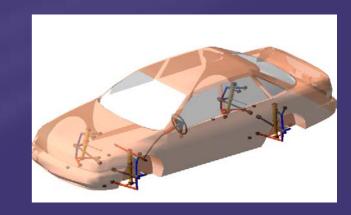


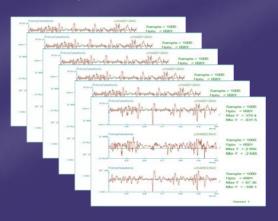


Data Handling Issues in Fatigue Analysis



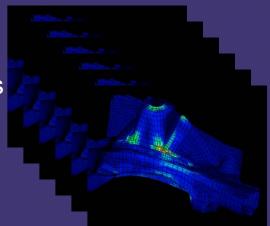
- Car body is subjected to more than 50 load channels (suspension connection points)
- Traditional approach requires
 50 unit load cases to be
 analyzed with FEA





Assign load channels to unit load cases units?

polarity?

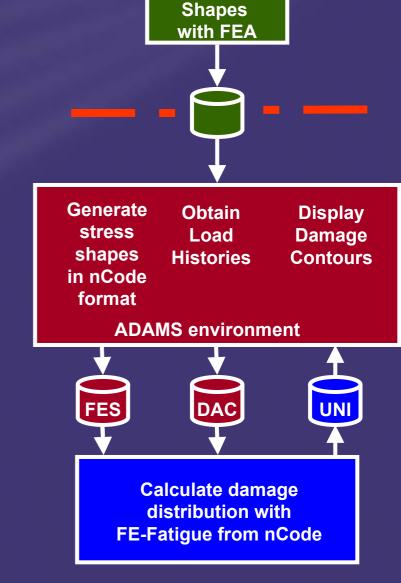


Issues: Not automated, error prone



Integrated Process for Fatigue Computation Generate Mode & Stress

- The MSR and Fatigue Toolkit
 - Minimizes data transfer between FEA, ADAMS, and FE-Fatigue
 - Generates consistent set of information
- Possible user error is minimized
- Durability process is automated

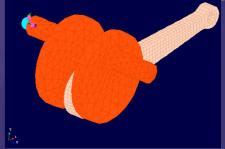


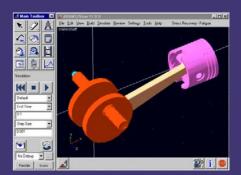


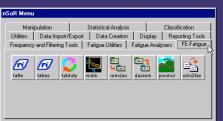
Integrated Process for Fatigue Computation (cont.)

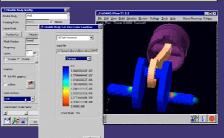


- Mesh components
- Step Two:
 - Run FEA to generate mode and stress shapes
 -> data is written to the MNF-file
- Step Three:
 - Extract the stress shapes from the MNF-file to nCode's FES-file
- Step Four:
 - Run ADAMS to obtain the component load histories (in terms of Modal Coordinates) and write the modal coordinates into DAC file
- Step Five:
 - Run FE-Fatigue to calculate fatigue damage on all nodes
- Step Six:
 - Run ADAMS/PPT to display components' damage contours













(none)

Velocity ICs

✓ default

▼ LBRF

Modal ICs

Flexible Body

Damping Ratio

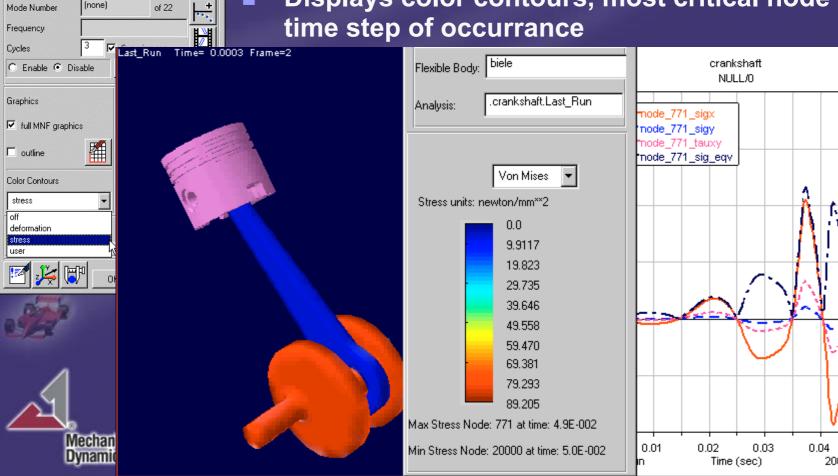
Datum Node

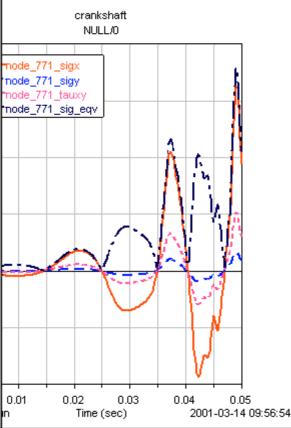
Position ICs

Example Process: Modal Stress Recovery



- **Computes and plots Stress Components and VonMises Invariant**
- Displays color contours, most critical node and time step of occurrance







Modal Stress Recovery Example 1/5



Start ADAMS and load MSR-Fatigue Plug-In

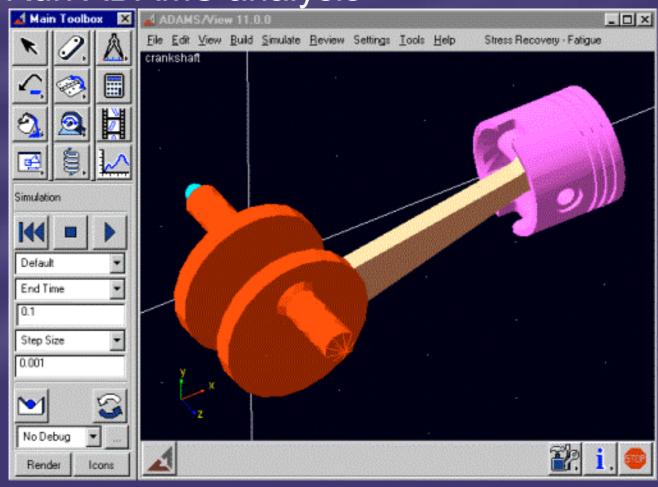






Modal Stress Recovery Example 2/5

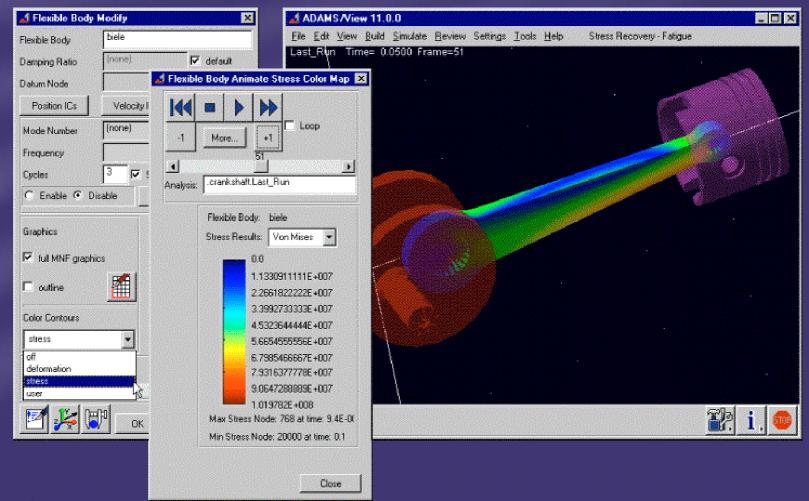
Run ADAMS analysis





Modal Stress Recovery Example 3/5

Animate Stresses on flex bodies





Stresses are computed for display purposes only, not stored as result sets



Modal Stress Recovery Example 4/5

 Compute and store stress/strain components at selected nodes

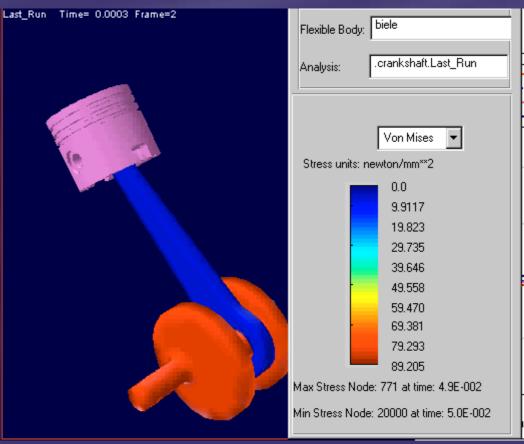
Compute Nodal Str	ess Components			×
Flexible body				
Analysis Node to add to select list Selected Nodes list				
☑ sigma Von Mises	□ sigma κ □ tau xy	□ sigma y □ tauyz	□ sigma z □ tau zx	
		0K	Apply	Cancel

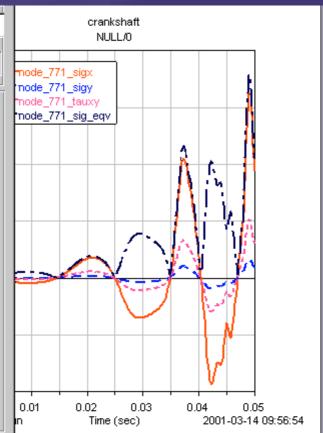




Modal Stress Recovery Example 5/5

Plot nodal stresses

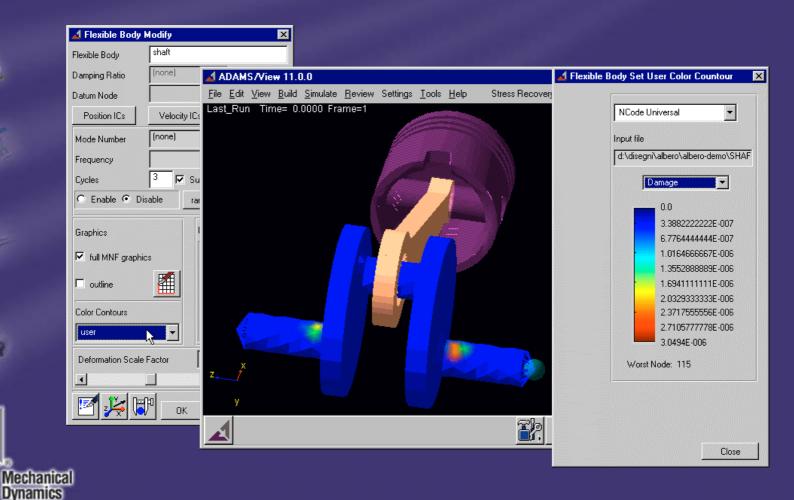






Example Process: Durability Analysis with FE-Fatigue

Computes and displays nCode FLP results within ADAMS/PPT



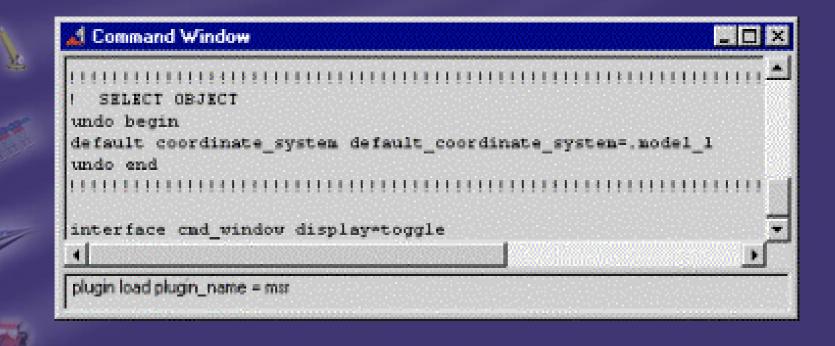


FE-Fatigue Example 1/10





Start ADAMS and load MSR-Fatigue Plug-In



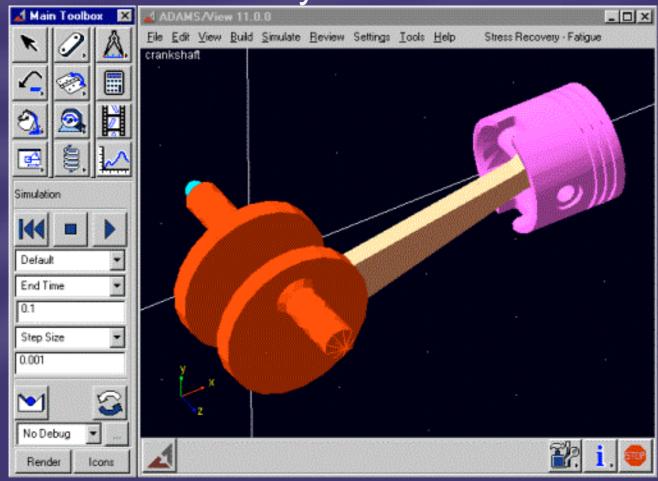




FE-Fatigue Example 2/10



Run ADAMS analysis

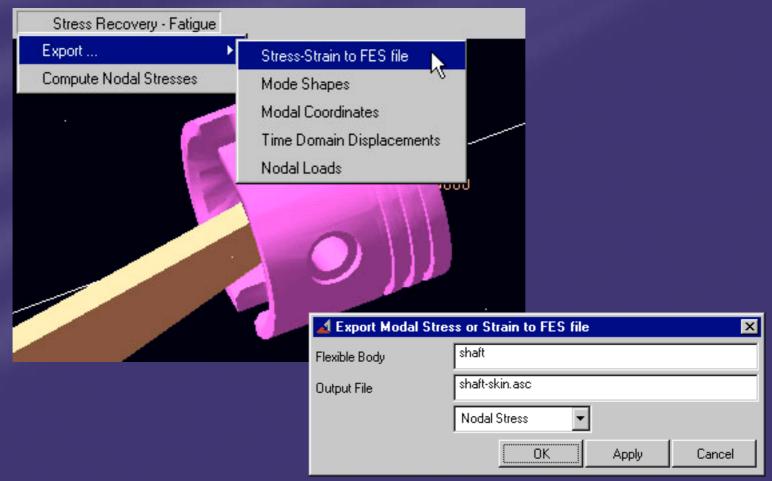




FE-Fatigue Example 3/10



Export stress shape function to partial FES file

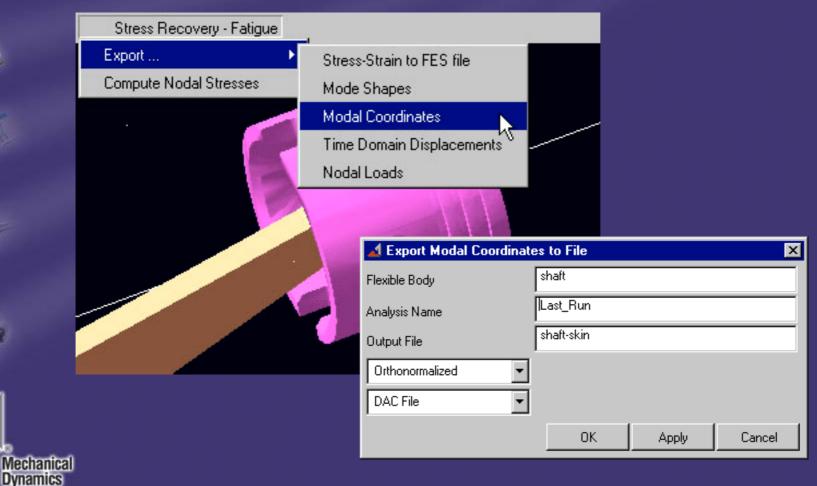




FE-Fatigue Example 4/10



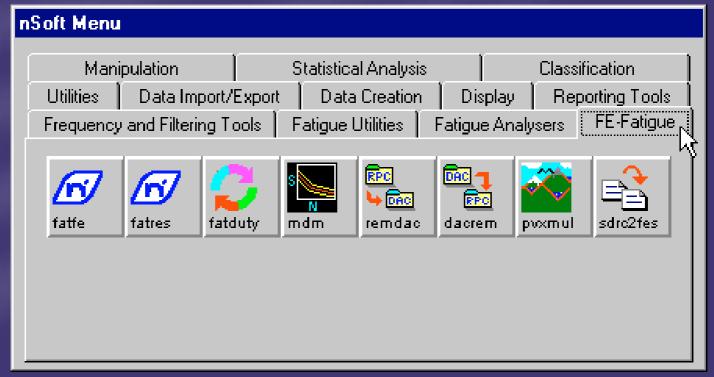
Export load histories (in terms of Modal Coordinates) to DAC files





FE-Fatigue Example 5/10

 Start nCode software, set working directory, and select FE-Fatigue task







FE-Fatigue Example 6/10



Specify analysis type

Ø FATFE - Partial to Full FES Completion ■ 🔣 🗷				
Job Name	SHAFT-SKIN			
Description	job string 1			
Analysis Type	S-N Analysis	T		
Stress Units	Pascals	•		
Time step deta type	Elastic	¥		
Advanced Options	O Yes			
✓ OK 🔀 Cancel		? Help		

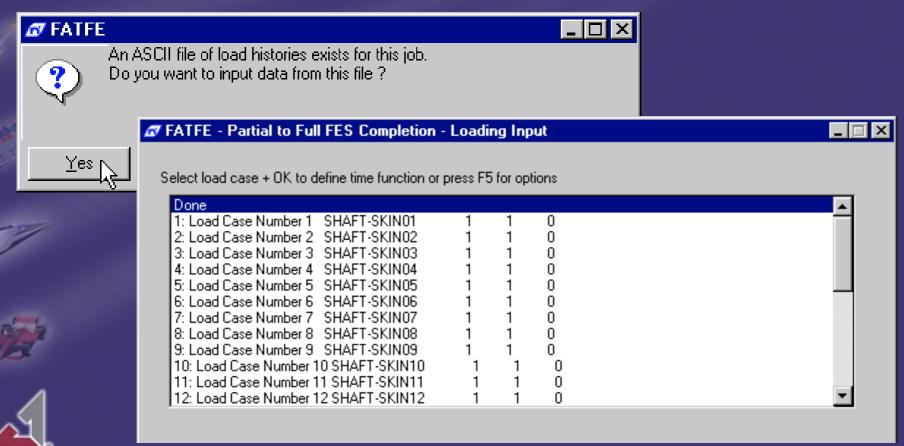




FE-Fatigue Example 7/10



Use output from previous ADAMS run as loading input





FE-Fatigue Example 8/10



Select material from database

FATFE - Partial to Full FES Completion	Material Input		_
Select Material/Group + OK or press £5 for options			
Gioto 3			
Method Select	Group 1		
Material name	Strength reduction (Kf)	1	
SAE5210_517_H SAE8630_254_NORM SAE8640_361_QT SAE9262_260_NORM	Surface Finish	No finish	•
SAE9262_271_QT sra_60	Surface Treatment	No treatment	Ŧ
sra_70 st00 unsg10200	Scale factor	1	
✓ OK Cancel		3	Help

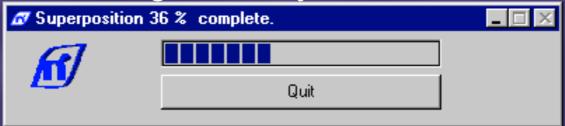




FE-Fatigue Example 9/10



Run Fatigue Analysis



Node	Damage	Life (Repeats)	Ave. Ratio	S.D. Ratio
115	3.0494E-6	3.279E5	0.208	0.311
194	3.0026E-6	3.33E5	0.037	0.183
503	1.9107E-6	5.234E5	0.048	0.122
427	4.8285E-7	2.071E6	0.14	8.25E-3
186	4.5523E-7	2.197E6	-0.0201	0.254
495	2.1068E-7	4.746E6	0.0291	0.152
187	1.735E-7	5.764E6	0.0693	0.122
89	1.5845E-7	6.311E6	0.246	0.214
401	1.5845E-7	6.311E6	0.24	9.63E-3
88	9.74E-8	1.027E7	0.198	0.0787

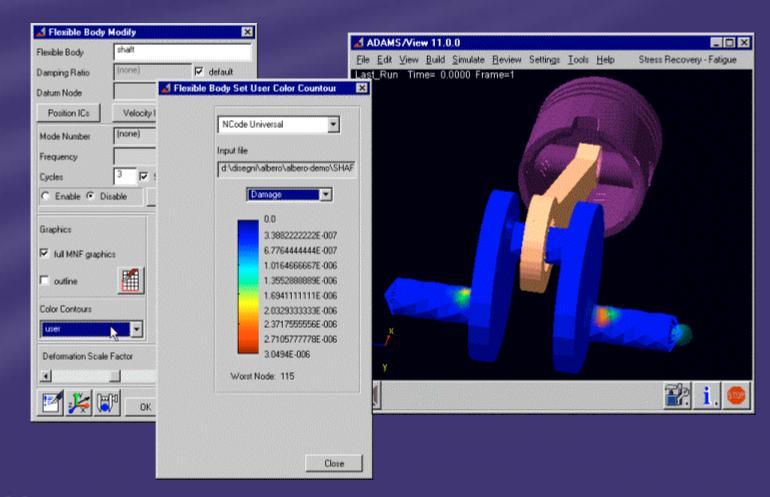




FE-Fatigue Example 10/10



Display Damage Contours in ADAMS/PPT





Supported Configurations





- Operating systems
 - Windows NT 4.0 and Windows 2000
 - ◆ IRIX 6.5 32 and 64 bit
- NASTRAN 70.5 and 70.7
- ANSYS 5.6 and 5.7
- FE-Fatigue 2.0







Summary



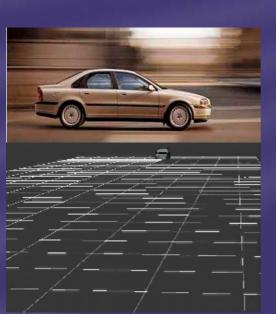
- The MSR and Fatigue Toolkit allows stress and fatigue evaluation within an integrated environment
- Switching between different tools is reduced to a minimum
- Process improvement in terms of efficiency and quality
- The Modal Stress Recovery Toolkit is available free of charge for all ADAMS/Durability users
 - preview of 12.0 functionality



Integrated Durability Process



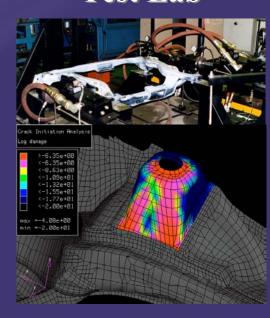




Full Vehicle **Test Lab**



Component **Test Lab**



- Seamless transition
 - from test track to test labs
 - from virtual to real
- nCode, MDI, and MTS are available to accelerate the integration process





Appendix









Principles of Modal Stress Recovery









Principles of Modal Stress Recovery

Assuming that the reduction of the full set of Mode Shapes of the Flexible Body to a sub-set is correct and comprehensive of all the required effects, the Stress Distribution related to the body deformation can be calculated as:

Stress Mode Shapes Method

$$\{\sigma\} = [\Phi_{\sigma}] \cdot \{p\}$$

$$\left[\phi_{\sigma} \right] = \left[\left\{ \phi_{\sigma} \right\}_{1}, \dots, \left\{ \phi_{\sigma} \right\}_{P+S} \right]$$

ortho-normalized Modal Stress Matrix

 $\{\phi_{\sigma}\}_{\dots P+S}$ (P=number of Normal Constrained Modes S = number of Static Correction Modes)

...P+S (P=number of Normal Constrained Modes S = number of Static Correction Modes)

Vector of Modal Coordinates

 $\sigma \}$ Stress Component Matrix





Principles of Modal Stress Recovery

- FE Calculation
 - Component Mode Synthesis, Craig Bampton
 - Normal Modes
 - Static Correction Modes
 - Residual Vectors for Distributed/Inertial/Thermal Loads
 - Modal Stress (and/or) Strain Tensors
 - .mnf file gene¥ration
 - Flexible Bodies in ADAMS
 - Correct Mechanism Dynamics
 - Correct Internal Deformations in the Flexible Component
 - Correct Boundary Loads and Constraints
 - Modal Coordinates
- Stress/Strain Computation

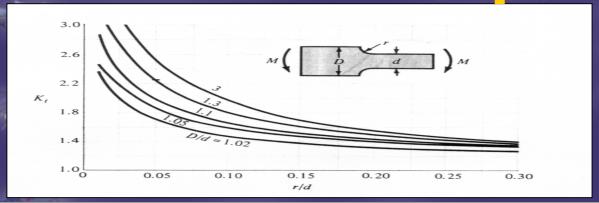
Dynamics

- Combining Modal Coordinates with Modal Precomputed (Stress/Strain) Tensors
- Post Processing in the most popular FE graphic programs and in ADAMS
 - Stress/Strain Time History
 - Stress/Strain Animation

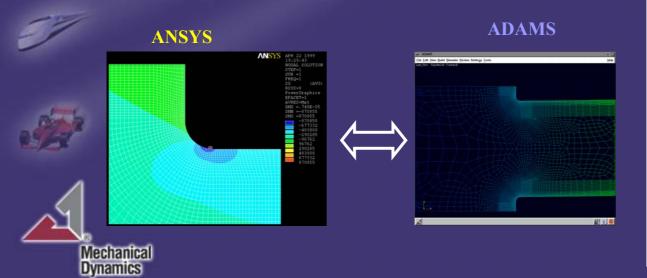


Principles of Modal Stress Recovery -

Example



Material	Young Modulus	D	d	r	s	M
	(N/m²)	(m)	(m)	(m)	(m)	(N m)
Steel	2.1E11	5.0E-2	3.85E-2	2E-3	1.0E-2	1



	Stress	K_t
ANSYS Node 408	887 580	2.192
ANSYS Node 4040	404 870	
ADAMS Node 408	873 400	2.164
ADAMS Node 4040	403 600	