

A GENERAL PROCEDURE FOR FINITE ELEMENT MODEL CHECK AND MODEL IDENTIFICATION

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ABSTRACT

Dynamic loads analysis using finite element models is a major task in the structural design process. An error free model to predict accurate loads or responses is essential for designing a vehicle to meet its performance requirements. A systematic approach employing MSC/NASTRAN direct matrix algorithm program is developed to check the residual loads and the rigid body strain energy for identifying improper modeling. The modal effective mass of each system mode is also computed using rigid body modes for statically determinate structures or constraint modes for statically indeterminate structures to identify the significant modes of the structure with respect to the constrained interfaces. These significant modes are the target modes to be identified in the payload structural qualification modal survey for generating a test-verified dynamic model used in the verification loads analysis. Furthermore, the modal kinetic energy distribution per degree of freedom and the kinetic energy fraction for each superelement are computed to identify the important subsystem local modes in addition to the standard strain energy fraction provided by MSC/NASTRAN. An example is presented to demonstrate this systematic approach for analytical model check and model identification.

INTRODUCTION

Dynamic loads analysis using finite element models is a major task in the structural design process. An error free model to predict accurate loads or responses is essential for designing a vehicle to meet its performance requirements. Several techniques, such as a 1-G check, enforced displacement check, and Cholesky check are available to verify analytical models. However, these techniques are applied at the analysis set level (i.e. at the A-set level in terms of MSC/NASTRAN terminology) and are unable to identify the problem area at the upper stiffness assembly levels. In 1985, an MSC/NASTRAN multi-level rigid body strain energy check routine [1] was developed to provide a means for checking an analytical model at three model stiffness assembly levels. For large complicated structures which are modeled as superelements, this routine needs to be extended in order to pin-point the modeling problems for each superelement and to identify the important dynamic characteristics of the structures.

Several additional techniques to detect modeling problems which result from improperly applied constraints and stiffness distribution as well as the algorithms to identify the dynamic characteristics of large complex dynamic models have been developed for MSC/NASTRAN version 67.5 structural superelement analysis. The residual loads and the rigid body strain energy concept are applied on the three levels of the stiffness matrices associated with MSC/NASTRAN analysis sets to identify the modeling errors. The total weight of the model and the weight of each up-stream superelement is calculated for comparison with the Mass Property Report. The modal effective mass [2] of each system mode is computed using the rigid body modes for statically determinate structures or the constraint modes for statically indeterminate structures. The modal effective mass is used widely to identify the significant modes of the structure with respect to the interfaces [3,4]. These significant modes are the target modes to be identified during the structural qualification modal survey. The accumulated modal effective mass is usually used to establish the frequency range of interest for predicting the dynamic responses using the mode transformation approach. Furthermore, the modal kinetic energy distribution per degree of freedom [5] and the kinetic energy fraction for each superelement are computed to identify the important subsystem local modes in addition to the standard strain energy fraction provided by MSC/NASTRAN.

An example using the Version 67.5 application direct matrix abstraction program (DMAP) is presented in this paper to demonstrate this systematic approach for analytical model check and model identification.

MODEL CHECK

Several techniques, such as a 1-G check, enforced displacement check, and Cholesky check are available to verify the analytical model. However, these techniques are applied at the analysis set level and are unable to identify the problem area at the upper levels. The multi-level rigid body strain energy check provides a means to check the model thoroughly. The basic concepts and algorithm which have been coded in the DMAP presented are discussed in the following sections.

Residual Loads and Residual Displacements Check

MSC/NASTRAN is well laid out in the various analysis sets: the G-set level contains all the degrees of freedom (DOF's) of the model; the N-set contains all the DOF's after removing from the G-set all the dependent DOF's due to multiple-point-constraint equations (MPC's) and rigid elements; the F-set contains all the DOF's after the single-point-constraint equations (SPC's) have been removed from the N-set. The basic principle applied in the residual loads and the residual displacements (separation ratio) check technique uses the

rigid-body modes $[\Phi_{rb}]$ to post-multiply the stiffness matrix $[K_s]$ of the model for determining the residual constraint forces $[P_{grnd}]$ associated with the rigid-body motion as shown in Eq. (1),

$$[K_s][\Phi_{rb}] = [P_{grnd}] \quad (1)$$

Note that the rigid body modes $[\Phi_{rb}]$ are determined from the structural geometry and have nothing to do with the rigid body modes computed in the later eigensolution process. Naturally, an ideal unconstrained model without errors would not have any constraint force associated with the rigid-body motion.

The residual displacement, $[S_{ratio}]$, in a specific direction is defined as the normalized displacement with respect to the stiffness in that direction with the effects of the other DOF's ignored as shown in Eq. (2),

$$[S_{ratio}] = \frac{[P_{grnd}]}{[Diag(K_s)]\{1\}^T} \quad (2)$$

where

- is the element-by-element division operator
- {1} is a unit column vector with 1's in all rows
- Diag() is a column vector containing the terms from the diagonals of the matrix enclosed in the parentheses

To minimize the numerical round-off error caused by dividing two small numbers in Eq. (2) the diagonal terms of $[K_s]$ less than a pre-selected stiffness threshold (currently set to 10^{-5}) are reset to zero. By computing the residual loads and the residual displacements at each unique level, the type of the modeling problems and the DOF's associated with it can be identified.

In general, the residual loads and the residual displacements at each DOF should be small (typically $<10^{-5}$ for translational DOF's and $<10^{-3}$ for rotational DOF's) for an error-free model. Therefore, a relatively large value at different levels indicates certain modeling errors. In addition, comparison of the residual load and the residual displacement at the same DOF may provide some indication of the numerical round-off problem. The possible modeling problems at each level are summarized as follows:

- G-level:
 - Coordinate coupling, e.g. RBE of grid points with different output coordinates
 - Ill-conditioning due to short beams
 - Element coordinate for pin flag DOF's offsets from grid point output coordinate
 - CELAS element between non-coincident points
- N-level:
 - Improper MPC equations
- F-level:
 - Over-constrained due to SPC's
 - Normal rotational stiffness is non-zero for CQUAD4 or CTRIA3 elements while the parameter K6ROT is turned on

It is emphasized that the model is GROUNDED (or CONSTRAINED) only if non-zero values not in the G-set or the N-set occur in the F-level. Any non-zero values in the G-level or the N-level indicate geometry modeling error or numerical round-off error.

Multi-Level Rigid-Body Strain Energy Check

The multi-level rigid body strain energy check is a similar technique as that of the residual displacement check. It simply determines the strain energy of the model associated with the rigid-body motion at different levels,

$$[K_{se}] = [\Phi_{rb}]^T [K] [\Phi_{rb}] \quad (3)$$

A large value (typically $>10^{-3}$ for translational direction and $>10^1$ for rotational direction) indicates modeling errors. The causes are the same as those summarized in the previous Section. The rigid body strain energy check is considered less important than the residual loads and the residual displacements check.

The residual displacements and the rigid-body strain energy computed by Eqs. (2) and (3) will be printed out for each superelement automatically. The residual loads of Eq. (1) will be printed out per user's request (parameter **PGPR**). The multi-level superelement approach to locate the possible modeling errors of smaller components is more efficient than the model check of the complete model.

Mass Property Check

The mass properties (total mass, inertia, and the location of the center of gravity) listed in the weight generation table of the MSC/NASTRAN output should agree with those specified in the updated mass property report. The weight table is generated for each superelement and can be used to verify the weight of tip superelements. If the superelement contains up-stream superelements the application DMAP will generate an additional weight table which includes the weights of all up-stream superelements. A special parameter (**WTONLY**) is available to perform the separation ratio and weight calculation without computing the frequencies of the components to minimize the computation cost.

Rigid-Body Frequency Check

The rigid body frequency check can be done on the residual superelement only. This is because the frequencies of the up-stream superelements are computed with respect to the boundary DOF's (similar to the Craig-Bampton modal model). However, the constrained normal modes computed for the up-stream superelement can be used to verify the minimum frequency requirement for that subsystem.

For an error-free unconstrained model, the first six modes should be rigid body modes. The frequencies of the six rigid-body modes in general should be five orders of magnitude less than the first elastic mode (the SUPORT entry should not be used in the Bulk Data when checking the rigid-body modes).

MODEL IDENTIFICATION

Determination of the structural fundamental frequency is a very important step in the design process. However, the identification of the mode shapes of a complex structure is always difficult. In addition, there is always a concern about how many modes should be kept if the mode transformation approach is to be used in determining the dynamic responses. The

effective mass which represents the contribution of each mode to the system response may be used as an indicator for mode selection.

Effective Mass Using Rigid Body Modes

The conventional approach is to compute the modal effective mass with respect to the rigid body modes $[\Phi_{rb}]$ assuming the load path can be ignored. This requires the calculation of the modal participation factors matrix (MPF) as defined in Eq. (4)

$$[MPF] = [\Phi_e]^T [M_s] [\Phi_{rb}] \quad (4)$$

where

$[\Phi_e]$ is the orthonormal mode shape matrix
 $[M_s]$ is the system mass matrix

Then the modal effective mass of each mode ($Meff_{rb} \%$) with respect to the rigid body modes can be obtained as follows:

$$[Meff_{rb} \%) = \frac{([\Phi_e]^T [M_s] [\Phi_{rb}]) \otimes ([\Phi_e]^T [M_s] [\Phi_{rb}])}{\{1\} [Diag([\Phi_{rb}]^T [M_s] [\Phi_{rb}])]^T} * 100 \quad (5)$$

where

\otimes is the element-by element multiplication operator
 $-$ is the element-by-element division operator
 $*$ is the multiplication operator
 $\{1\}$ is a unit column vector with 1's in all rows
 $Diag()$ is a column vector containing the terms from the diagonals of the matrix enclosed in the parentheses

Effective Mass Using Constraint Modes

On the other hand, if the structure is statically indeterminate then the modal effective mass computed using rigid body mode approach is incorrect because the interface load path has not been taken into account. In this case the constraint mode set instead of the rigid body mode set has to be used. The constraint mode $[\Phi_{cm}]$ is determined from Eq. (6),

$$[\Phi_{cm}] = \begin{bmatrix} -K_{ii}^{-1} K_{ib} \\ I_{bb} \end{bmatrix} \quad (6)$$

where

$[K_{ii}]$ is the stiffness corresponding to non-interface DOF's
 $[K_{ib}]$ is the stiffness corresponding to interface DOF's
 $[I_{bb}]$ is an identity matrix corresponding to interface DOF's

The modal effective mass of each mode ($Meff_{cm} \%$) with respect to the constraint modes can be obtained from Eq. (7),

$$[Meff_{cm} \%] = \frac{[(\Phi_e)^T [M_s] [\Phi_{cm}]) \otimes (\Phi_e)^T [M_s] [\Phi_{cm}])]}{[1] [Diag((\Phi_{cm})^T [M_s] [\Phi_{cm}]))^T]} * 100 \quad (7)$$

Ideally, if all the modes were included in the prediction of the dynamic response and the structure was linear, the total modal effective mass and inertia would be equal to the physical mass and inertia with respect to the reference point. In reality, the significant modes within the frequency range of interest are considered to have been included in the analysis if the ratio of the total modal effective mass to the physical mass is higher than 85%.

Two levels of modal effective masses, G-level and J-level, are computed for the superelement approach. The G-level modal effective mass, GMEFF%, includes the contribution from the up-stream superelements while the J-level modal effective mass, JMEFF%, is the contribution of that superelement only. By examining the modal effective mass of each mode in the translational and rotational directions, the motion of each mode can be identified and the significant modes of the structure can be selected. The total modal effective mass of the significant modes can be determined to check if they indeed contribute most to the dynamic responses of the structure. The calculation of modal effective mass is controlled by the parameter MEFF. The rigid body mode approach is the default method. The constraint mode approach is selected automatically by the DMAP if BSET cards defining the interface DOF's are used in the model.

Occasionally, the modal effective mass of a subsystem constrained at its interfaces is desired for validating the minimum frequency requirement for the secondary structure and for supporting the subsystem stand alone modal survey. A parameter, CBMEFF, can be turned on for each superelement to compute the modal effective mass corresponding to the component Craig-Bampton model.

Modal Kinetic Energy per DOF

The difficulty of mode identification of a dynamic model increases substantially as the model becomes larger and more complex. In order to assist the analyst in differentiating system modes and local modes, a concept of determining the modal kinetic energy per degree of freedom is developed. This modal kinetic energy check is based on the orthogonality principle as shown in Eq. (8),

$$[\Phi_e]^T [M_s] [\Phi_e] = [I] \quad (8)$$

where $[M_s]$ is the mass matrix of the system, $[\Phi_e]$ is the orthonormal mode shape matrix, and $[I]$ is the identity matrix.

Equation (8) can be expanded by using a special element-by-element multiplication operation and normalizing the diagonal terms to 100 to determine the percent modal kinetic energy distribution per DOF for each mode,

$$[KENG\%] = [\Phi_e] \otimes ([M_s] [\Phi_e]) \quad (9)$$

where \otimes represents the element-by-element multiplication operator. In the modal kinetic energy matrix, $[KENG\%]$, the rows represent the DOF's of the model and the columns represent the calculated modes. The value at any location of $[KENG\%]$ is the percent contribution a particular DOF makes to the total kinetic energy of that mode. The summation of any column should be equal to 100. The modal kinetic energy per DOF exposes the

specific directional behavior of the response. This is a remarkable improvement over standard NASTRAN element strain energy output which lumps all the DOF contribution for an element and does not distinguish between directional effects. The calculation of modal kinetic energy per DOF is activated by specifying the parameter **KE** in the bulk data or case control deck.

Similar to the effective mass calculation, two levels of kinetic energy per DOF, G-level and J-level, are computed for the superelement approach. The G-level kinetic energy per DOF, **KENG%**, includes the contribution from the up-stream superelements while the J-level, **JKENG%**, is the contribution of that superelement only. For superelement with up-stream superelements, the actual kinetic energy contribution of the DOF's are those listed in the **JKENG%**. The J-level kinetic energy per DOF, **JKENG%**, is also used widely in the industry to determine accelerometer locations for modal survey testing. Theoretically, the DOF's with higher kinetic energy respond more effectively to the dynamic environment. Consequently, the data collected at these locations are less biased and are better suited for use in modal parameter estimation.

Modal Kinetic Energy Fraction

The modal effective mass criteria used to identify the significant system modes is usually less sensitive for determining important local modes. To identify the significant local modes of a subsystem for improving the dynamic response prediction, the modal kinetic energy fraction of that subsystem, **[KEF]** is calculated. **[KEF]** is a measurement of the amount of system kinetic energy contained within each subsystem for any given mode. The modal kinetic energy fraction is computed based on Eq. (10),

$$[KEF] = \frac{[Diag([\Phi_c]^T [M_c] [\Phi_c])] }{[Diag([\Phi_s]^T [M_s] [\Phi_s])]} \quad (10)$$

where

- [\Phi_c]** is the component partition of the system mode shape matrix
- [\Phi_s]** is the system mode shape matrix
- [M_c]** is the component mass matrix
- [M_s]** is the system mass matrix
- is the element-by-element division operator
- Diag()** is a column vector containing the terms from the diagonals of the matrix enclosed in the parentheses

The modal kinetic energy fraction is usually employed in conjunction with the modal effective mass for selecting important modes. A rule of thumb is that if a mode has more than 50% of the modal kinetic energy of the whole system in any single subsystem, that mode is considered to be a significant local mode for that component. The calculation of modal kinetic energy fraction is controlled by the parameter **SEKEF**.

Modal Strain Energy Fraction

The modal strain energy fraction, **[SEF]**, is similar to **[KEF]** in its calculation. **[SEF]** is a measurement of how much of the strain energy of the system is in each subsystem for any given mode. The modal strain energy fraction is computed using Eq. (11),

$$[SEF] = \frac{[Diag([\Phi_c]^T [K_c] [\Phi_c])]}{[Diag([\Phi_s]^T [K_s] [\Phi_s])]} \quad (11)$$

where

- $[\Phi_c]$ is the component partition of the system mode shape matrix
- $[\Phi_s]$ is the system mode shape matrix
- $[K_c]$ is the component mass matrix
- $[K_s]$ is the system mass matrix
- is the element-by-element division operator
- Diag() is a column vector containing the terms from the diagonals of the matrix enclosed in the parentheses

The calculation of modal strain energy fraction is controlled by the NASTRAN standard parameter **SESEF**. A rule of thumb is that if a mode has more than 50% of the modal strain energy of the whole system in any single subsystem, that mode is considered to be a significant local mode for that component.

DMAP IMPLEMENTATION

The application of the model check and model identification DMAP is straightforward. The diagnostic information of the model will be computed by simply inserting the DMAP in the MSC/NASTRAN Executive Deck. The application DMAP is shown in Appendix A. Users can define the following parameters in the Case Control or Bulk Data to activate the options available in the DMAP.

Parameters	Default	Description
WTONLY	No	If this parameter is selected (PARAM,WTONLY,YES), only the model check and the weight generation procedures are executed. This parameter is very useful for checking a free-free model before the effective masses are computed for the constrained configuration.
BGPTB	No	Print coordinate systems and grid point locations in basic coordinate system.
PGPR	No	Print residual loads.
PGFILTER	1.0E-3	Suppress the printing of residual loads smaller than PGFILTER.
MEFF	No	Compute modal effective mass. Rigid body mode approach is the default method. Constraint mode approach will be automatically activated if the interface degrees of freedom are defined in the B-Set in the residual superelement.
LMEFF%	2.0	Generate a column vector, MODEID, with a value of 1.0 at the rows whose modal effective mass is larger than LMEFF%.
CBMEFF	No	Compute modal effective mass for subsystems constrained at interfaces.

KE	No	Compute modal kinetic energy per DOF.
KEFILTER	1.0	Suppress the printing of any modal kinetic energy per DOF (KENG%) less than KEFILTER.
SEKEF	No	Compute subsystem modal kinetic energy fraction.

EXAMPLE

A Space Station integrated truss segment, S3/S4, as shown in Figure 1 was used to demonstrate the application of the model check and model identification DMAP. The segment is supported by flexure assemblies at the trunnions and keels to simulate the National Space Transportation System (NSTS) flight boundary conditions. The interface DOFs are 60200X, 60200Z, 57526Y, 60100X, 60100Z, 232097Y, 232091Z, and 232094Z and the Orbiter coordinate system is used to identify the direction. The system frequencies and modal effective mass, up to 50 Hz, for the segment S3/S4 are computed and compared for the rigid body mode approach and the constraint mode approach.

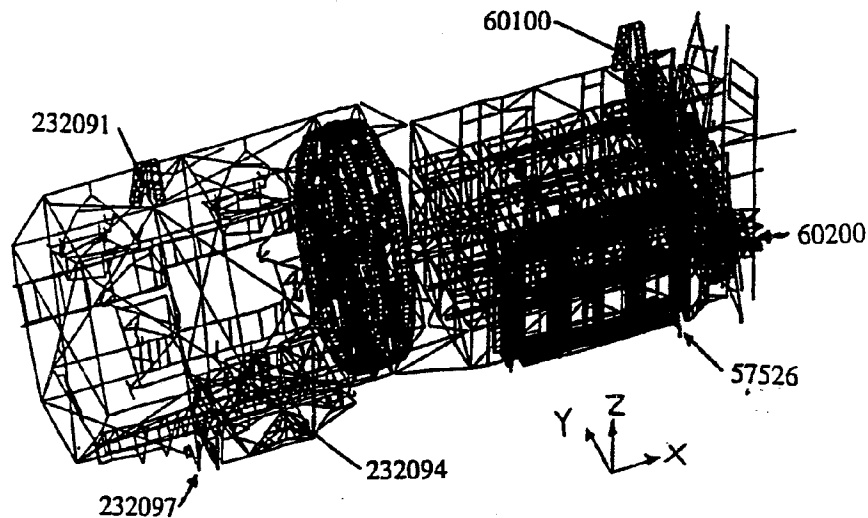


Figure 1. Segment S3/S4 Supported at Flexure Assembly

For the rigid body mode approach there are always six columns of modal effective mass, three translational and three rotational, associated with each mode. Table 1 shows the modal effective mass of the target modes (translational modal effective mass equal to or greater than 2%, or the kinetic energy fraction or the strain energy fraction greater than 50%) of segment S3/S4 using the conventional rigid body mode approach. Fourteen target modes are identified. The primary X mode has 78.14% modal effective mass at 8.36 Hz, the primary Y mode has modal effective mass 86.47% at 6.67 Hz, and the primary Z mode has 40.92% modal effective mass at 9.58 Hz. The total modal effective masses in the X-axis, Y-axis, and Z-axis are 95.56%, 93.17%, and 76.90%, respectively. The total modal effective mass in the Z direction is lower than that in the X or Y direction because more constraints (primary and secondary trunnions) are imposed in that direction.

KE	No	Compute modal kinetic energy per DOF.
KEFILTER	1.0	Suppress the printing of any modal kinetic energy per DOF (KENG%) less than KEFILTER.
SEKEF	No	Compute subsystem modal kinetic energy fraction.

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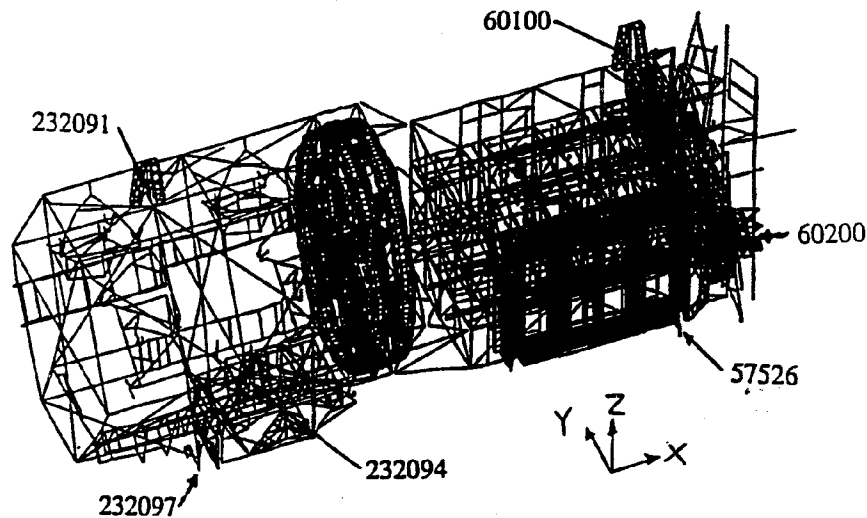


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On the other hand, Table 2 shows the modal effective mass computed from the constraint mode approach where the load path is taken into account. Each mode has a modal effective mass value corresponding to the constrained degree of freedom. For this example, which is constrained at eight interface degrees of freedom, there are eight columns of modal effective mass for each mode. The energy of each mode is transmitted differently to the interface based on the load path. However, the total modal effective mass of the target modes in the same vibratory direction at different interface locations should converge to a similar value. For this example X direction has 94.44% and 93.13%, Y direction has 97.02% and 92.36%, and Z direction has 82.04%, 81.50%, 84.59% and 85.21%. A total of twenty-one target modes are identified for S3/S4 using the constraint mode approach. The primary X mode has a maximum of 66.30% modal effective mass at 8.36 Hz, the primary Y mode has a maximum of 91.95% modal effective mass at 6.67 Hz, and the primary Z mode has a maximum of 41.15% modal effective mass at 6.67 Hz. The averaged total modal effective masses in X-, Y-, and Z-axis are about 94%, 94%, and 83%, respectively. It can be seen that the modal effective mass distribution for the target modes has been changed significantly and more target modes are selected for the constraint mode approach. The constraint mode approach is the recommended approach for computing the modal effective mass and selecting the target modes for statically indeterminate structures because it represents the actual flight conditions.

CONCLUSIONS

A systematic approach to identify the modeling errors and the dynamic characteristics of a structural model is developed and coded in DMAP for MSC/NASTRAN implementation. The modeling of each superelement is checked in three analysis levels such that improper modeling can be easily located. Two methods to select the significant structural modes for dynamic load prediction are included in the DMAP to accommodate different attachment schemes. The modeling and the dynamic behavior of a structural mathematical model can be assessed and fully understood by incorporating this DMAP.

REFERENCES

- [1] Parker, G. R. and Brown, J. J., "DMAP Alter for Performing Basic Constraint Checks on Structural Modeling," MSC/NASTRAN User's Conference, March 1985, Pasadena, CA.
- [2] Trubert, M., "Assessment of Galileo Modal Test Results for Mathematical Model Verification," 25th AIAA SDM conference, paper no. 84-1066, April 1984, pp 528-541.
- [3] Osman, E. A. & Krausser, D. G., "A Generalized Form for the Modal Effective Mass," 63rd Shock & Vibration Symposium, Las Cruces, NM, October, 1992.
- [4] Chung, Y. T. & Sernaker, M. L., "Assessment of Target Mode Selection Criteria for Payload Modal Survey," Proceeding of the 12th International Modal Analysis Conference, Jan. 1994, Honolulu, Hawaii.
- [5] Parker, G. R. and Brown, J. J., "Kinetic Energy DMAP for Mode Identification," MSC/NASTRAN User's Conference, March 1982, Pasadena, CA.

Appendix A

Model Checkout and Model Identification DMAP

```

$ $
$ $ PRINT BCPDT AND CSTM TABLES *
$ $ *****
$ $ *****
$ $
$ $ COMPIL SUBDMAP=PHASED SOUIN=MSCSOU,NOLIST,NOREF
$ $ ALTER 206
$ $ TYPE PARM,,CHAR3,Y,(BGFTB=NO ) $
$ $
$ $ PVT PVT,CASES/PVTS/ $
$ $ IF (BGFTB=YES) THEN $
$ $ TABPRT BGPDTS//BGPDY $
$ $ TABPRT CSTM//CSTM $
$ $ ENDIF $ BGFTB=YES
$ $
$ $ *****
$ $ * GROUNDING CHECK - COMPUTE SEPARATION RATIO *
$ $ *****
$ $
$ $ *** ADJUST WEIGHT GENERATION TABLE IF MODAL MODEL ARE ***
$ $ *** USED FOR UP-STREAM SEES AND/OR RESIDUAL SE ***
$ $
$ $ *** NOTE: U6 SET (USET OR SEUSET) SHOULD BE DEFINED FOR ***
$ $ *** GRIDS IN DMIG REPRESENTING THE COMPONENT MODES ***
$ $
$ $ COMPIL SUBDMAP=SEM61, SOUIN=MSCSOU,NOREF,NOLIST
$ $ ALTER 11,12
$ $
$ $ COMPIL SUBDMAP=SEKR SOUIN=MSCSOU,NOLIST,NOREF
$ $ ALTER 50
$ $ TYPE PARM,NDL,I,Y,GRDPT $
$ $ TYPE PARM,RS,Y,WTMASS $
$ $ VEC USET/V1,U6SET/G/COMP/TU6 $
$ $ PARTN MH V1,U6SETG/MINUSQ,41 $
$ $ MERGE MMINUSQ,V1,U6SETG/MINUSQ,1 $
$ $ GRPWG BGPDTS,CSTMS,EQEXINS,MJIN,OOQ/OGPWGTSE/GRDPT/WTMASS $
$ $ OGPWGTSE// $
$ $
$ $ ALTER 129 $ CRAY
$ $ TYPE DB,PVT $
$ $ TYPE PARM,,CHAR3,Y,(PGRP=NO ) $
$ $ TYPE PARM,NDL,I,N,ZUZRI $
$ $ TYPE PARM,RS,Y,(PGFILTER=0.001) $
$ $ VECPLT,,BGPDTS,EQEXINS,CSTMS,/RBGTRANS/GRDPT//4 $ RIGID BODY MODES
$ $ TRNSP,,RBGTRANS/RBGQ $GLOBAL RIGID BODY MODES : G-SIZE ROWS X 6 COLS
$ $
$ $ REMOVE GENERALIZED COORDINATES (SEQSET) FROM RIGID-BODY MODES, IF ANY
$ $ VEC USET/V1,QSET/G/COMP/TQ/ $
$ $ ADD V1,QSETG/V1,QSET/VGMINUSQ $
$ $ IF (NOUP>0) THEN $
$ $ DBVIEW KLAADUP=KLA4(WHERE SEID=*, AND WILDCARD=TRUE) $
$ $ DBVIEW MAPSUP=MAPS(WHERE SEID=*, AND WILDCARD=TRUE) $
$ $ SEMA EQEXINS,SLIST,EMAP,,KLAADUP,MAPSUP/KLAAG/SEID/
$ $ LUSSETS/QUAL=SEID/0 $
$ $ DIAGONAL KLAAG/VQUPSE/COLUMN/1. $
$ $ ADD VGINUSQ,VQUPSE/VGNOO $
$ $ EQUIVX VGN00/VGMINUSQ/ALWAYS $
$ $ ENDIF $ NOUP>0

```

```

PARTN RBGQ,VGMINUSQ/RBGNOQ,1 $
MERGE RBGNOQ,,,,,VGMINUSQ/RBG/1 $
$ $
$ $ SAVE RBG IN DATA BASE FOR FUTURE USE IN OTHER SUBDMAP
ZUZRI = SEID $
EQUIVX RBG/ZUZR01/ALWAYS $
$ $
$ $ PVT CASES/PVTS/ $
$ $
$ $ COMPUTE G-LEVEL SEPARATION RATIO
MPYAD KGG RBG,/PGRNDG $ COMPUTE GROUNDING FORCES
DIAGONAL KGG/KGGD/COLUMN/1 $
MATMOD KGGD/KGGD/2//1,1,0E-5 $
PARAML KGGD/TRAILER/2/S,N,GSIZE $
MATGEN, /VNONE/6/FSIZE/0/FSIZE $
ADD VNONE,KGGD/KGGD//2 $
PARAML KGGD/TRAILER/5/S,N,NZWD $
IF (NZWD>0) THEN $
MATMOD KGGD,,,,,KGGDINV,28 $
MPYAD KGGDINV/PGRNDG,/SRATIOG $ G-LEVEL SEPARATION RATIOS
IF (PGRP=YES) THEN $
MATGPR GPLS,USET,SILS,PGRNDG//H7/C//PGFILTER $
ENDIF $ (PGRP=YES)
MATGPR GPLS,USET,SILS,SRATIOG//H7/C//1,E-5 $
ENDIF $ NZWD>0
$ $
$ $ COMPUTE N-LEVEL SEPARATION RATIO
VEC USET/NLINK/G/COMP/N. $ PUT 1'S IN G-SET FOR N-SET DOFS
PARTN RBG,NLINK/RBN,40 $GET N-SET ROWS OF RIGID BODY MATRIX
MPYAD KNN,RBN,/PGRNDN $ COMPUTE GROUNDING FORCES
DIAGONAL KNN/KNNDX/COLUMN/1 $
MATMOD KNNDX,,,,,KNND/2//1,1,0E-5 $
PARAML KNND/TRAILER/2/S,N,NSIZE $
MATGEN, /VNONE/6/NSIZE/0/NSIZE $
ADD VNONE,KNND/KNND//2 $
PARAML KNND/TRAILER/5/S,N,NZWD $
IF (NZWD>0) THEN $
MATMOD KNND,,,,,KNNDINV,28 $
MPYAD KNNDINV/PGRNDN,/SRATION $ N-LEVEL SEPARATION RATIOS
IF (PGRP=YES) THEN $
MATGPR GPLS,USET,SILS,PGRNDN//H7/N//PGFILTER $
ENDIF $ (PGRP=YES)
MATGPR GPLS,USET,SILS,SRATION//H7/N//1,E-5 $
ENDIF $ NZWD>0
$ $
$ $ COMPUTE F-LEVEL SEPARATION RATIO
VEC USET/FLINK/G/COMP/F $
PARTN RBG,FLINK/RBF,40 $
MPYAD KFF,RBF,/PGRNDF $
DIAGONAL KFF/KFFDX/COLUMN/1 $
MATMOD KFFDX,,,,,KFFD,2//1,1,0E-5 $
PARAML KFFD/TRAILER/2/S,N,FSIZE $
MATGEN, /VNONE/6/FSIZE/0/FSIZE $
ADD VNONE,KFFD/KFFD//2 $
PARAML KFFD/TRAILER/5/S,N,NZWD $
IF (NZWD>0) THEN $
MATMOD KFFD,,,,,KFFDINV,28 $
MPYAD KFFDINV/PGRNDF,/SRATIOF $ F-LEVEL SEPARATION RATIO
IF (PGRP=YES) THEN $
MATGPR GPLS,USET,SILS,PGRNDF//H7/F//PGFILTER $
ENDIF $ (PGRP=YES)
MATGPR GPLS,USET,SILS,SRATIOF//H7/F//1,E-5 $
ENDIF $ NZWD>0
$ $
$ $ COMPUTE G-, N-, AND F-LEVEL STRAIN ENERGY

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```

ELSE IF (NOTSET=1) THEN $
EQUIVX KLAAG/MEAXX/ALWAYS $
ELSE $
ADD5 KAA,KLAA,,,MKAAX $
ENDIF $(NOQSET=1)
VEC USE7/VIBSETA/A/COMP/TB $
PARTN MMAAX,VIBSETA,KILKIB,MBB/-1 $
SOLVE KILKIB/PHICM/I/1 $
MERGE PHICM,IDENB,,,VIBSETA/PHICMA/1 $
ZUZR=0 $
EQUIVX PHICMA/ZUZR04/ALWAYS $
SDRI USE7,PHICMA,,GOAX,GM,KFS,KSS/
PHICMG,/I/APPI $
ELSE $
***** RIGID BODY MODE APPROACH
*****
***** RIGID BODY MODE APPROACH
*****
$ RETRIEVE/GENERATE RIGID-BODY MODES
$ ZUZR1=SEID $
$ DBVIEW RBGX=X=ZUZR01 (WHERE ZUZR1=SEID) $
$ $
$ PARAML RBGX//PRESENCE////S,N,NRBGG $
$ IF (NRBGG=>0) THEN $
$ COPY RBGX/RBGG/ALWAYS $
$ $
$ ELSE $
PARAML USE7/USE7////S,N,LUSETS $
GENERATE RIGID-BODY MODES FOR RESTART RUN
VECPLOT,,BGPDTS,EQEXNS,CSTMS,,/RBGTRANS/GRDPNT/4 $ RIGID BODY MODES
TRNSP,,RBGTRANS/RBGG $GLOBAL RIGID BODY MODES : G-SIZE ROWS X 6 COLS
$ REMOVE GENERALIZED COORDINATES (SEQSET) FROM RIGID-BODY MODES, IF ANY
VEC USE7/VGMINUSQ/G/COMP/Q/ $
IF (RSONLY) THEN $
ELSE $
IF (NOUP>0) THEN $
DBVIEW KLAAG/MEAXX/ALWAYS $
DBVIEW MAPSUP=MAPS(WHERE SEID=* AND WILDCARD=TRUE) $
SEMA EQEXNS,SLIST,EMAP,,KLAAG,MAPSUP/KLAAG/SEID/
DIAGONAL KLAAG/VQUPSE/COLUMN/1. $
ADD VGMINUSQ,VQUPSE/VGNOQ $
EQUIVX VGNOQ/VGMINUSQ/ALWAYS $
ENDIF $(NOUP=0)
ENDIF $(RSONLY)
PARTN RBGQ,,VGMINUSQ/RBGNQ,,/1 $
MERGE RBGNQ,,,,VGMINUSQ/RBGG/1 $
$ $
$ ENDF $(NRBGG=>0)
ENDF $(SEID=0 AND (NRSET=0 AND NASET>NBSET))
ENDF $(NPHICMA=>0)
$ GET NO OF B SET DOF FOR EFFECTIVE MASS CALCULATION USING
$ CONSTRAINT MODE APPROACH
PARAML PHICMG/TRAILER/1/S,N,NRSBSET $
IF (SEID=0 AND (NRSET=0 AND NASET>NBSET)) TABPRT USE7/EQEXNS/
/USE7/0/2097152 $
$ $

```

```

IF (RSONLY) THEN $
$ ELSE $
IF (NOUP>0) THEN $
$ CHECK THE PRESENCE OF THE PHYSICAL MASS MATRIX (MIJ)
IF (NOMIJ >= 0) THEN $
$ *****
$ * EFFECTIVE MASS WITH CONTRIBUTION OF UP-STREAM SE'S
$ * (FOR RESIDUAL SE OR SE WITH UP-STREAM SUPERELEMENTS)
$ * *****
$ * GMPF <- MODAL PARTICIPATION FACTOR WITH UP-STREAM SE'S
$ * GMPFMODE <- EFFECTIVE MASS PER MODE WITH UP-STREAM SE'S
$ * GMEHFSUM <- TOTAL EFFECTIVE MASS WITH UP-STREAM SE'S
$ * GMEFF% <- PERCENT EFFECTIVE MASS PER MODE WITH UP-STREAM
SE'S
$ * TGMEFF% <- PERCENT TOTAL EFFECTIVE MASS W/O UP-STREAM SE
$ * *****
$ COMPUTE MODAL PARTICIPATION FACTOR (GMPF) AND MODAL
EFFECTIVE MASS (GMEHFSUM) FOR UP-STREAM SE'S
MPYAD UG,MGG/UGMGX/1 $
IF (SDBC=NO) THEN $
MPYAD UGMGX,PHICMG/GMPF $
ELSE $
MPYAD UGMGX,RBGG/GMPF $
ENDF $(SDBC=NO) $
$ COMPUTE MODAL EFFECTIVE MASS FOR EACH MODE (GMEFFMOD).
DIAGONAL GMEHFSUM/WHOLE/2/0 $
TRNSP GMEFFSQ/GMODE/ITEM $
ADD GMODETEM/GMEFFMOD/CMASSWT/1 $
$ COMPUTE TOTAL EFFECTIVE MASS AND INERTIA OF THE SYSTEM (GMEFFSUM),
AND CONVERT THEM TO WEIGHT UNIT.
TRNSP GMPF/GMPPT $
MPYAD GMPPT,GMPF/GMEFFSQ $
DIAGONAL GMEFFSQ/GMSUMTEM/COLUMN/1/0 $
ADD GMSUMTEM/GMEFFSUM/CMASSWT/1 $
$ COMPUTE % EFFECTIVE MASS OF EACH MODE
$ EXTRACT PHYSICAL MASS AND INERTIA FROM WEIGHT TABLE
DBVIEW MAAPUC=MAA(WHERE SEID=* AND WILDCARD=TRUE) $
DBVIEW MAPSUP=MAPS(WHERE SEID=* AND WILDCARD=TRUE) $
SEMA EQEXNS,SLIST,EMAP,,MJJ,MAADPG,MAPSUP/MGGYTX/
SEID/LUSETS/SEID/UPFM $
GPWG BGDPTS,CSTMS,EQEXNS,MGGYTX/OGPWX/GRDPNT/WTMASS $
$ GENERATE 6X1 VECTOR FOR PHYSICAL MASS AND INERTIA
WTRW=6 $
MATGEN,,GWTSUM/7/6/1 $
DO WHILE(WTRW>0) $
WTRW=WTRW-1 $
WTRW=6-WTRW $
MATGEN,,/V1W/6/WTRW/INDEX/1 $
PARAML OGPWX/(WTRW/2/WTRW/D/S,N,WTDATA $
WTRW/ALUB=CMPX(WTDATA,00) $
ADD V1WT/V2WT/WTRW/VALUE/ $
ADD V2WT/GWTSUM/V3WT/ $
EQUIVX V3WT/GWTSUM/ALWAYS $
ENDDO $

```



```

ENDIF $(NULLGOQ > 0)
VEC USE/VIOTOG/COMP/VO $
MERGE GOOX_VIOTOG/GGOX/1 $
$ RENORMALIZE COMPONENT MODES TO COMPONENT MASS MATRIX
SMPYAD GGOXI,MGG,GGOXI,,,MGGSCB/3//1 $
DIAGONAL MGGSCB/MCSQR/TSQUARE/4.5 $
MPYAD GGOXI,MCBSQR/1,GGOX $
$
$ COMPUTE EFFECTIVE MASS OF C-B MODEL ELASTIC MODES
MPYAD GGOX,MGG/GQMGX/1 $
MPYAD GQMGX,REBG/CBMEF $
DIAGONAL CBMEF/CBMEFSQD/WHOLE/2.0 $
TRANSP CBMEFSQD/CBMODE/TE $
ADD CBMODE/TE/CBMEF/MOD/CMASSTW/1 $
TRANSP CBMEF/CBMEF/1 $
MPYAD CBMEF/CBMEF/CBMEFSQ $
DIAGONAL CBMEFSQ/CBMSUM/TE/COL/UMN/1.0 $
ADD CBMSUM/TE/CBMEFSUM/CMASSTW/1 $
ADD CBMEF/MOD/TSUM/MAT/1/CBMEF/1.0 $
ADD CBMEFSUM/TSUM/MAT/1/CBMEF/1.0 $
$
$ RETRIEVE COMPONENT FREQUENCIES
LAMX,CMLAMA/CBLMAT/1 $
MATMOD CBLMAT,,,/CBFREQ/1/3 $
MATPRN CBFREQ// $
$
$ PRINT EFFECTIVE MASS PER MODE, TOTAL EFFECTIVE MASS,
$ PERCENT EFFECTIVE MASS PER MODE, AND PERCENT TOTAL
MESSAGE //EFFECTIVE MASS PER MODE OF C-B COMPONENT MODES/ $
MATPRN CBMEF/MOD/ $
MESSAGE //TOTAL EFFECTIVE MASS OF C-B COMPONENT MODES/ $
MATPRN CBMEFSUM// $
MESSAGE //PERCENT EFFECTIVE MASS PER MODE OF C-B COMPONENT/ $
MATPRN CBMEF// $
MESSAGE //PERCENT TOTAL EFFECTIVE MASS OF C-B COMPONENT/ $
MATPRN TCBMEF// $
ENDIF $(RSONLY)
ENDIF $(CBMEF= YES)
$
$ *****
$ * COMPUTE THE MODAL KINETIC ENERGY PER DEGREE *
$ * OF FREEDOM FOR UP-STREAM SUPERELEMENT (MODE *
$ * SHAPES ARE NORMALIZED TO MASS MATRIX) *
$ *****
$
$ *** CHECK IF CALCULATION OF KINETIC ENERGY PER
$ *** EACH DEGREE OF FREEDOM IS REQUESTED
IF (KE= YES) THEN $
$
DIAGONAL KGG/VGSET/COLUMN/0.0 $
IF (RSONLY) THEN $
ELSE $
IF (NOUTP > 0) THEN $
$
$ CHECK THE PRESENCE OF THE PHYSICAL MASS MATRIX (MUJ)
PARAMI MUJ/PRESENCE///S,N,NOMIT $
IF (NOMIT > 0), THEN $
$ COMPUTE MODAL KINETIC ENERGY PER DOF (KEDOF) AND CHECK
$ GENERALIZED MASS (MSTAR)
MPYAD MGG/UG/MPEH/1 $
ADD UG,MPEH/KEDOF//1 $
MPYAD VGSET,KEDOF/MSTAR/1 $
MATPRN MSTAR// $
$

```

```

$
ADD IMEFFMOD,ISUM/MAT/IMEFF/((100.0,0.0)/2 $
ADD IMEFFSUM,JWTSUMX/IMEFF/((100.0,0.0)/2 $
$
$ GENERATE MODE INDEX FOR MODES HAVING TRANSLATIONAL EFFECTIVE
$ MASS HIGHER THAN LMEFF% (DEFAULT IS 2%)
IF (SDBC=NO) THEN $
MATMOD IMEFF,,,/IMEFF/M/2///LMEFF $
MATMOD J/ONE/6/NR/SBSET/INTRSBSET $
ELSE $
MATMOD IMEFF,,,/IMEFF/M/2///LMEFF $
MATGEN //XZYEFF/6/6/3/3 $
MATGEN //VONE/6/3/0/3 $
PARTN IMEFFS,IVXZYEFF/IMEFF/M/1 $
ENDIF $(SDBC=NO)
MPYAD J/ONE/IMEFF/M,JWTMODES/1 $
DIAGONAL JWTMODES/IMEFF/WHOLE/0.0 $
ZUZRI=SEID $
EQUIVX IMODEID/ZUZRO/3/ALWAYS $
MESSAGE //TARGET MODE INDEX FOR SE W/O UP-STREAM SE/ $
MATPRN IMODEID// $
$
$ PRINT EFFECTIVE MASS PER MODE, TOTAL EFFECTIVE MASS,
$ PERCENT EFFECTIVE MASS PER MODE, AND PERCENT TOTAL
MESSAGE //EFFECTIVE MASS PER MODE W/O UP-STREAM SE/ $
MATPRN IMEFF/MOD// $
MESSAGE //TOTAL EFFECTIVE MASS W/O UP-STREAM SE/ $
MATPRN IMEFFSUM// $
MESSAGE //PERCENT EFFECTIVE MASS PER MODE W/O UP-STREAM SE/ $
MATPRN IMEFF// $
MESSAGE //PERCENT TOTAL EFFECTIVE MASS W/O UP-STREAM SE/ $
MATPRN TMEFF// $
ENDIF $ NOMIT >= 0
$
ENDIF $(MEFF= YES)
$
$ *****
$ * EFFECTIVE MASS OF COMPONENT W.R.T. ITS INTERFACES *
$ *****
$ * CBFREQ : CANTILEVERED FREQUENCY MATRIX FOR C-B COMPONENT *
$ * CBMEF : MODAL PARTICIPATION FACTOR FOR C-B COMPONENT *
$ * CBMEFSUM : EFFECTIVE MASS PER MODE FOR C-B COMPONENT MODES *
$ * CBMEF% : TOTAL EFFECTIVE MASS PER C-B COMPONENT MODES *
$ * CBMEF% : PERCENT EFFECTIVE MASS PER MODE FOR C-B COMPONENT *
$ * TCBMEF% : TOTAL PERCENT EFFECTIVE MASS PER C-B COMPONENT *
$ *****
$
$ ***** COMPUTE EFFECTIVE MASS OF COMPONENT (UPSTREAM SE) *****
$ ***** CONSTRAINED AT INTERFACES (C-B MODEL EFFECTIVE MASS) *****
IF (CBMEF= YES) THEN $
IF (RSONLY) THEN $
ELSE IF (SEID=0) THEN $
ELSE $
VEC USE/VIANOB/A/COMP/VO $
PARTN GOAQ,VIANOB/L,GOQXN/1 $
MATMOD GOQXN,,,VIGOQ/1/2/S,N, NULLGOQ $
IF (NULLGOQ > 0) THEN $
PARTN GOQXN,VIGOQ,GOQXN/1 $
ELSE $
COPY GOQXN/GOQX/ALWAYS $

```

```

$ $ DETERMINE THE LOCATION AND DIRECTION OF MAX KINETIC ENERGY FOR A
$ $ MODE, AND ITS PERCENT CONTRIBUTION TO THAT MODE.
MATMOD KEDOF, /KEDOFMAX, /1 $
MATMOD KEDOFMAX, /DKAKEMX, /28 $
NORM KEDOF/KENORM $
ADD KENORM/KENORM%(100.0,0.0) $
MATMOD KENORM, /SCALEMAX, /2//0.9999 $
MPYAD SCALEMAX, /DKAKEMX, /KEMAX $
ADD KEMAX, /KEMAX%(100.0,0.0) $
$ MATGPR GFLS, /KEMAX%/H/G//KEFILTER $
$ $ COMPUTE PERCENT CONTRIBUTION ON KINETIC ENERGY FOR EACH DOF
MPYAD KENORM, /DKAKEMX, /KENG/ $
ADD KENG, /KENG%(100.0,0.0) $
MESSAGE //KINETIC ENERGY PER DEGREE OF FREEDOM WITH UP-STREAM SE/ $
IF (KENORMPR= YES) THEN $
MESSAGE //KENORMPR= YES) THEN $
MESSAGE //KINETIC ENERGY PER DOF NORMALIZED TO MAX KE WITH SE / $
MATGPR GFLS, /KENG%/H/G//KENORM% $
ENDIF $ (KENORMPR= YES)
$ $ CHECK THE TOTAL KINETIC ENERGY OF THE UP-STREAM SE
MPYAD VGET, /KENG%, /KESUMUP/1 $
MATPRN KESUMUP// $
ENDIF $ NMODS=0
ENDIF $ RONLY
$ $ **** COMPUTE MODAL KINETIC ENERGY PER DOF (KEDOF) AND CHECK
$ $ **** GENERALIZED MASS (MSTAR) W/O CONTRIBUTION OF UP-STREAM SE
$ $ ****
$ $ CHECK THE PRESENCE OF THE PHYSICAL MASS MATRIX (MIJ)
IF (NOMIJ >= 0) THEN $
MPYAD MIJ, /UG//MPHI/ $
ADD UG, /MIJ//KEDOF//1 $
MPYAD VGET, /KEDOF//MSTAR/1 $
$ MATPRN IMSTAR// $
$ $ DETERMINE THE LOCATION AND DIRECTION OF MAX KINETIC ENERGY FOR A
$ $ MODE, AND ITS PERCENT CONTRIBUTION TO THAT MODE.
MATMOD IKEDOF, /IKEDOFMX, /7 $
NORM IKEDOF/KENORM $
ADD IKEDOF/KENORM%(100.0,0.0) $
MATMOD IKENORM, /JSCALEMX, /2//0.9999 $
MPYAD JSCALEMX, /DKAKEMX, /JREMAX $
ADD JREMAX, /JREMAX%(100.0,0.0) $
$ MATGPR GFLS, /IKEMAX%/H/G//KEFILTER $
$ $ COMPUTE PERCENT CONTRIBUTION ON KINETIC ENERGY FOR EACH DOF
MPYAD IKENORM, /DKAKEMX, /JKENG/ $
ADD JKENG, /JKENG%(100.0,0.0) $
MESSAGE //KINETIC ENERGY PER DEGREE OF FREEDOM W/O UP-STREAM SE/ $
MATGPR GFLS, /IKEMAX%/H/G//KEFILTER $
IF (IKENORMPR= YES) THEN $
MESSAGE //KINETIC ENERGY PER DOF NORMALIZED TO MAX KE W/O SE/ $
MATGPR GFLS, /IKENORM%/H/G//IKENORM% $
ENDIF $ (IKENORMPR= YES)
$ $ CHECK THE TOTAL KINETIC ENERGY OF THE UP-STREAM SE
MPYAD VGET, /JKENG%, /JKESUMUP/1 $
MATPRN JKESUMUP// $
ENDIF $ NMODS=0
ENDIF $ KB= YES
$ $
$ $ ***** KINETIC ENERGY FRACTION FOR EACH SUPERELEMENT *****
$ $ COMPUTE KINETIC ENERGY FRACTION FOR EACH SUPERELEMENT
$ $ SEKEFA <- KINETIC ENERGY FRACTION OF TIP
$ $ SUPERELEMENTS
$ $ SEKEFG <- KINETIC ENERGY FRACTION W/O CONTRIBUTION
$ $ OF UP-STREAM SUPERELEMENTS
$ $ *****
$ $ COMPILER SUBDMAP=POSTREIG SOUIN=MSCSOULNOLIST,NOREF
ALTER 15
TYPE DR,MAA,MLAA,MIJ $
TYPE PARM, CHAR3,Y,(NASTIPS=NO) $
TYPE PARM, I,Y,(NASTAPE=99) $
TYPE PARM, CS,Y,CSEID $
TYPE PARM, CHAR3,Y,(SEKEF=NO) $
MATMOD LAMMAT,,,,/TKE, /1/4 $
$ $
ALTER 21,21
PVT PVT, CASEDR/PVTX/ $
$ $
ALTER 48
IF (SEKEF = YES) THEN $
IF (NOUP = -1) THEN $ KINETIC ENERGY FRACTION PER MODE
MESSAGE // DMAP INFORMATION MESSAGE 9046 (POSTREIG) -/
' THE FRACTION OF TOTAL KINETIC ENERGY IN EACH OF THE /
' RESIDUAL STRUCTURE MODES IN TIP SUPERELEMENT/SEID $
ADD MAA,MLAA/MAA $
MPYAD YMAA,PHLS,/FAM, $
ADD FAM,PHLS,/FUM//, $ EL,MPY
MATGEN /UAM6/NOASET/NOASET $ UNIT COLUMN, A-SIZE
MPYAD FUM,UAM,/SEKEA/1 $
ADD SEKEA,TKE/SEKEFA//2 $ EL, DIV.
MATPRN SEKEFA/$ KINETIC ENERGY FRACTION FOR TIP
ELSE $
MESSAGE // DMAP INFORMATION MESSAGE 9046 (POSTREIG) -/
' THE FRACTION OF TOTAL KINETIC ENERGY IN EACH OF THE /
' RESIDUAL STRUCTURE MODES IN NON-TIP SUPERELEMENT/SEID $
PARAML MIJ//PRESENCE///S,N,NOMIJ $
IF (NOMIJ > -1) THEN $
MPYAD MIJ,PHG,/FGM $
ADD FGM,PHG,/XGM//1 $
MATGEN /UGM/6/NOGSET/NOGSET $
MPYAD XGM,UGM,/SEKEG/1 $
ADD SEKEG,TKE/SEKEG//2 $
MATPRN SEKEG/$ KINETIC ENERGY FRACTION FOR NON-TIP
ENDIF $ (NOMIJ > -1)
ENDIF $ (SEKEF= YES)

```