





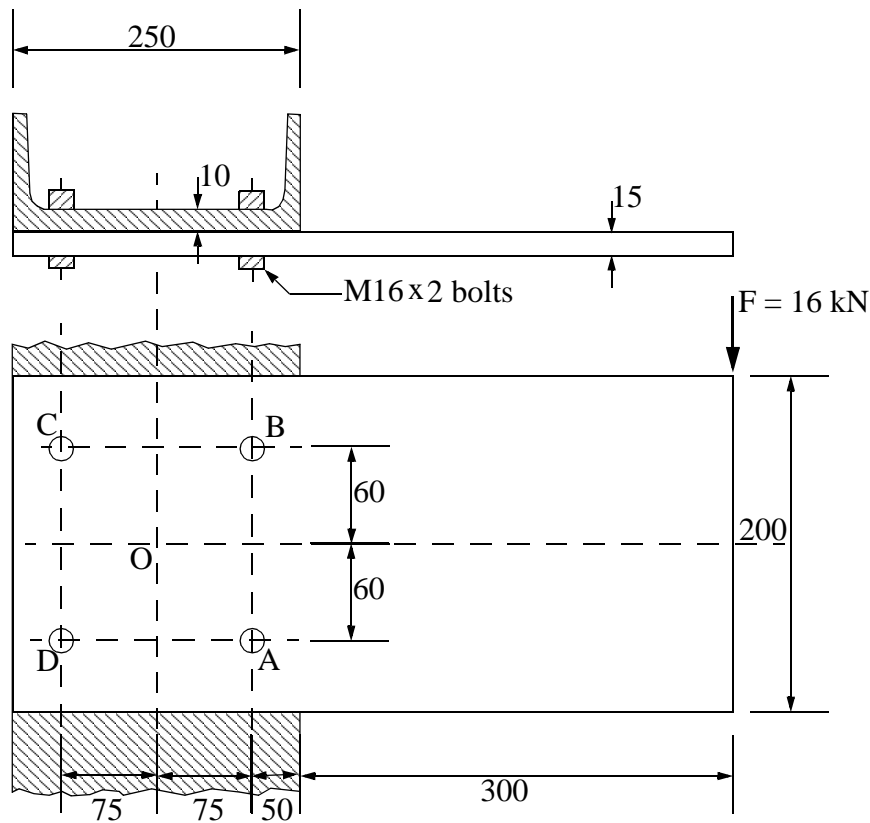
**Model Description:**

The goal of the example is to analyze the shear and moment reactions at the four bolts by using RBE3 elements, instead of drawing **Figure 10a10a.1**.

Force F and moments that exist on **Figure 10a.1** will be applied at point O.

Below in **Figure 10a.1** is a diagram of a rectangular steel bar cantilevered to a steel channel using four bolts. There is an external 16 kN load applied, what is the resultant load on each bolt.

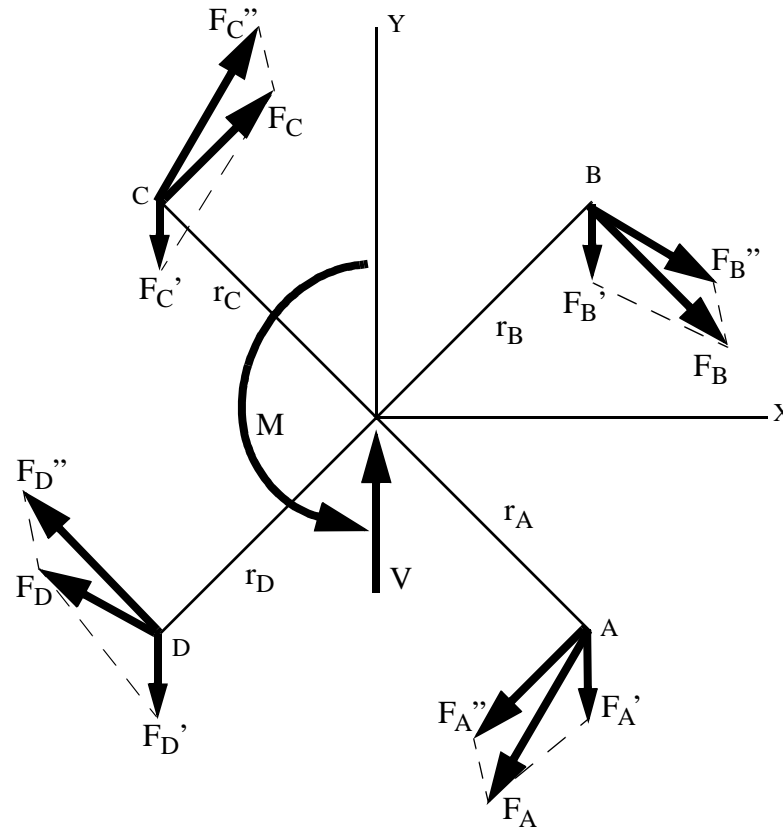
**Figure 10a.1** - Diagram and Dimension of Bolts and Fixture



**Table 10a.1** - Values for Bolt Model

<b>Outer Radius</b>	<b>2 mm</b>
<b>Inner Radius</b>	<b>1.9 mm</b>
<b>Elastic Modulus</b>	<b>7.1E10 N/mm<sup>2</sup></b>
<b>Poisson's Ratio</b>	<b>0.3</b>

**Figure 10a.2** - Diagram of Force Componets



**Solution**

Point  $O$ , the centroid of the bolt group in **Figure 10a.2**, is found by symmetry. If a free-body diagram of the beam were constructed, the shear reaction  $V$  would pass through  $O$  and the moment reaction  $M$  would be about  $O$ . These reactions are:

$$V = 16kN \qquad M = 16(425) = 6800Nm$$

In **Figure 10a.2**, the bolt group has been drawn to a larger scale and the reactions are shown. The distance from the centroid to the center of each bolt is:

$$r = \sqrt{(60)^2 + (75)^2} = 96.0mm$$

The primary shear load per bolt is:

$$F'' = \frac{V}{n} = \frac{16}{4} = 4kN$$

Since the secondary shear forces are equal, the calculation comes to:

$$F''' = \frac{Mr}{4r^2} = \frac{M}{4r} = \frac{6800}{4(96.0)} = 17.7kN$$

---

## Suggested Exercise Steps:

- Generate a geometry model of the four bolts and create finite element for each bolt.
- Create node 999 to represent point O.
- Define material (MAT1) and element (PORP1) properties.
- Apply the fixed boundary constraints on each bolt. Create shear, and moment reactions at node 999.
- Idealize a rigid end connecting node 999 and one end of each bolt with rigid elements (RBE2).
- Prepare the model for linear static analysis (SOL 101).
- Generate an input file and submit it to the MSC.Nastran solver for normal modes analysis.
- Review the results.

### Exercise Procedure:

1. Users who are not utilizing MSC.Patran for generating an input file should go to Step 16 otherwise, proceed to Step 2.

2. Create a new database called **bolt\_load.db**

**File/New Database**

*New Database Name*

**bolt\_load**

**OK**

In the *New Model Preference* form set the following. :

*Tolerance*

◆ **Default**

*Analysis code:*

**MSC/NASTRAN**

**OK**

3. Activate the entity labels by selecting the **Show Labels** icon on the toolbar.



**Show Labels**

4. Create curves to represent the bolts.

◆ **Geometry**

*Action:*

**Create**

*Object:*

**Curve**

*Method:*

**XYZ**

*Vector Coordinate List:*

**<0 0 10>**

*Auto Execute*

*Origin Coordinates List:*

**[75 -60 0]**

**Apply**

Repeat Step 3, changing *Origin Coordinates List* to **[75 60 0]**, **[-75 60 0]**, and **[-75 -60 0]**.

*Origin Coordinates List:*

**[75 60 0]**

**Apply**

*Origin Coordinates List:*

**[-75 60 0]**

**Apply**

*Origin Coordinates List:*

**[-75 -60 0]**

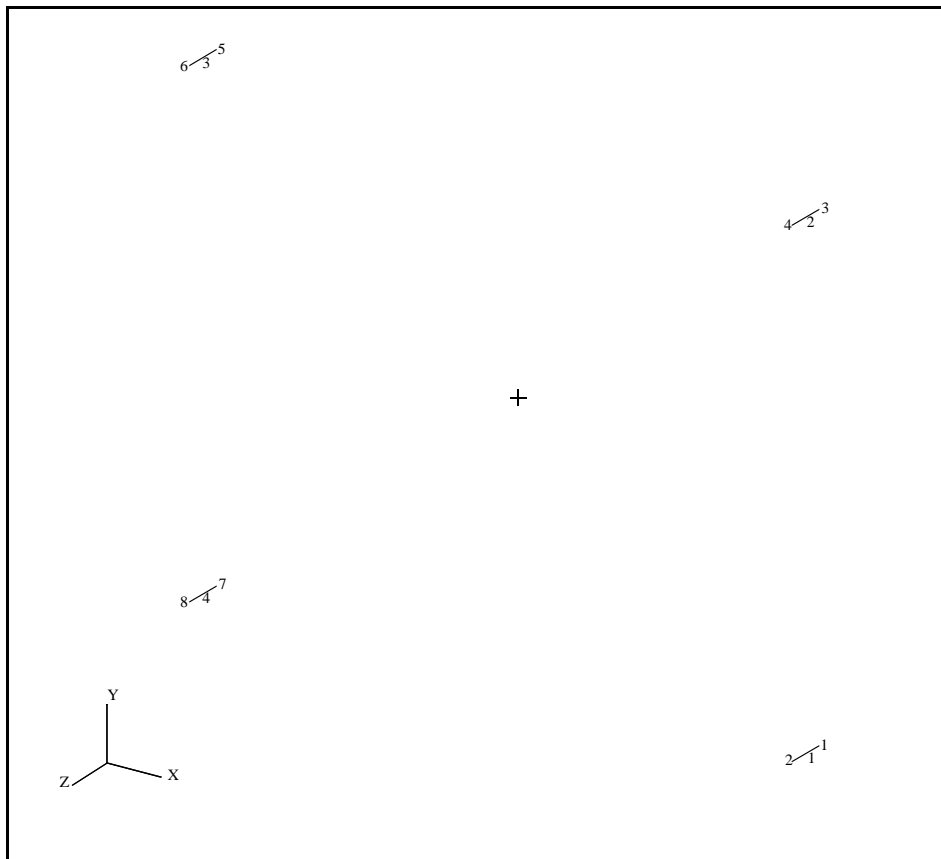
**Apply**

To see the curves that was just created, change the view to **Iso 1 View** by selecting on this icon:



**Iso 1 View**

**Figure 10a.3** - Geometry Model Of the Bolts



5. Create the finite element model and mesh the curves.

◆ **Finite Elements**

<i>Action:</i>	<input type="text" value="Create"/>
<i>Object:</i>	<input type="text" value="Mesh"/>
<i>Type:</i>	<input type="text" value="Curve"/>
<i>Global Edge Length:</i>	<input type="text" value="10"/>
<i>Curve List:</i>	<input type="text" value="Curve 1:4"/>
<input type="text" value="Apply"/>	

6. Node create node 999 and add to model.

◆ **Finite Elements**

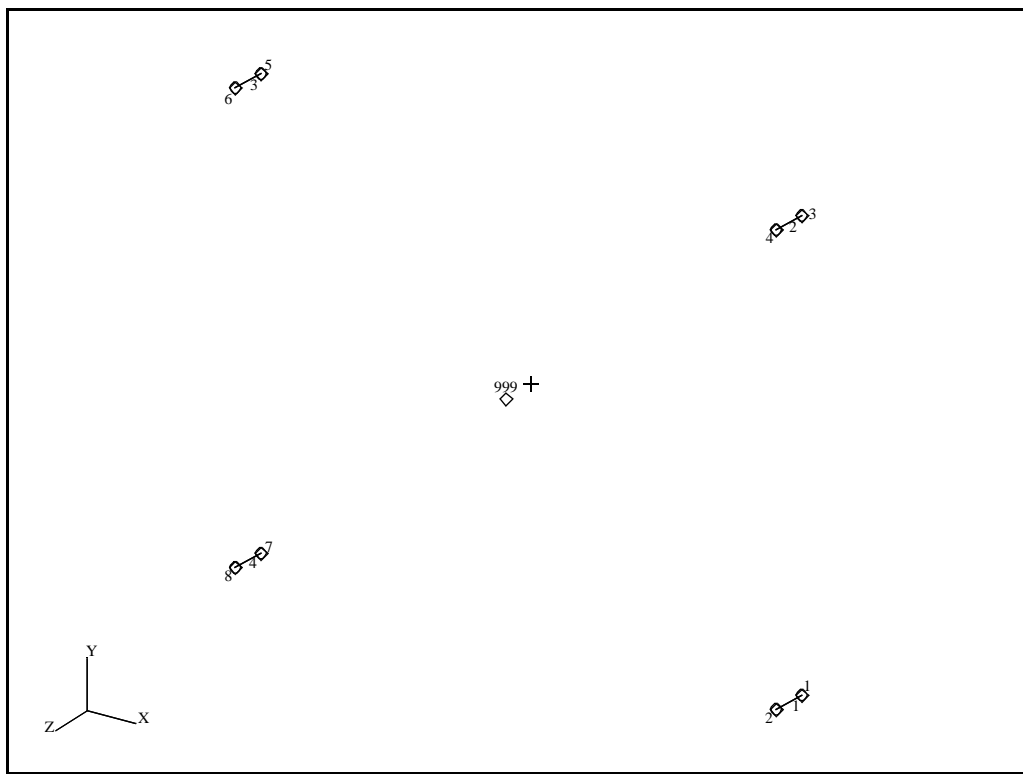
<i>Action:</i>	<input type="text" value="Create"/>
<i>Object:</i>	<input type="text" value="Node"/>
<i>Type:</i>	<input type="text" value="Edit"/>
<i>Node ID List:</i>	<input type="text" value="999"/>
<input type="checkbox"/> <i>Associate with Geometry</i>	
<input type="checkbox"/> <i>Auto Execute</i>	
<i>Node Location List:</i>	<input type="text" value="[0 0 0]"/>
<input type="text" value="Apply"/>	

To see the node, node 999, use **Node Size** by clicking on this icon:



**Node Size**

**Figure 10a.4** - Geometry, Finite Elements, and Node 999



7. Now you will create the material properties for the plate.

◆ **Materials**

*Action:*

**Create**

*Object:*

**Isotropic**

*Method:*

**Manual Input**

*Material Name*

**mat\_1**

**Input Properties ...**

*Elastic Modulus*

**7.1E10**

*Poisson Ratio*

**0.3**

**Apply**

**Cancel**

8. Give the curves a 3D shape by using Properties.

◆ **Properties**

<i>Action:</i>	<input type="text" value="Create"/>
<i>Dimension:</i>	<input type="text" value="1D"/>
<i>Type:</i>	<input type="text" value="Beam"/>
<i>Property Set Name</i>	<input type="text" value="Prop_1"/>

**Input Properties ...**

<i>Material Name</i> <small>(Select from <b>Material Property Sets</b> box)</small>	<input type="text" value="m:mat_1"/>
<i>Bar Orientation:</i> <small>note: both lines are for Bar Orientation</small>	<input type="text" value="Node 999"/> <input type="text" value="Node Id"/>

■ **Associate Beam Selection**

Click the beam library icon:



<i>New Section Name:</i>	<input type="text" value="Prop_1"/>
--------------------------	-------------------------------------

Choose **Tube Section:**



<i>R1</i>	<input type="text" value="2.0"/>
<i>R2</i>	<input type="text" value="1.9"/>

<i>Select Members</i>	<input type="text" value="Curve 1:4"/>
-----------------------	--

---

**Apply**

9. Next, apply the boundary conditions to the model.

◆ **Loads/BCs**

*Action:*

**Create**

*Object:*

**Displacement**

*Type:*

**Nodal**

*New Set Name*

**Fixed\_ends**

**Input Data...**

*Translations <T1 T2 T3>*

**<0, 0, 0>**

*Rotations <R1 R2 R3>*

**<0, 0, 0>**

**OK**

**Select Application Region...**

*Geometry Filter*

◆ **Geometry**

*Curve List*

**Point 1:7:2**

*(See Figure D.3)*

**Add**

**OK**

**Apply**

10. Now create the force on node 999.

◆ **Loads/BCs**

*Action:*

**Create**

*Object:*

**Froce**

*Type:*

**Nodal**

*New Set Name*

**Shear\_Reaction**

**Input Data...**

*Forces <F1 F2 F3>*

**<0, -16, 0>**

**OK**

Select Application Region...

*Geometry Filter*

*Curve List*

*(See Figure D.4)*

Add

OK

Apply

◆ FEM

Node 999

11. Apply the moment to the model.

◆ Loads/BCs

*Action:*

Create

*Object:*

Force

*Type:*

Nodal

*New Set Name*

Moment\_Reaction

Input Data...

*Forces <F1 F2 F3>*

<0, 0, 0>

*Moment <M1 M2 M3>*

<0, 0, 6800>

OK

Select Application Region...

*Geometry Filter*

◆ FEM

*Curve List*

Node 999

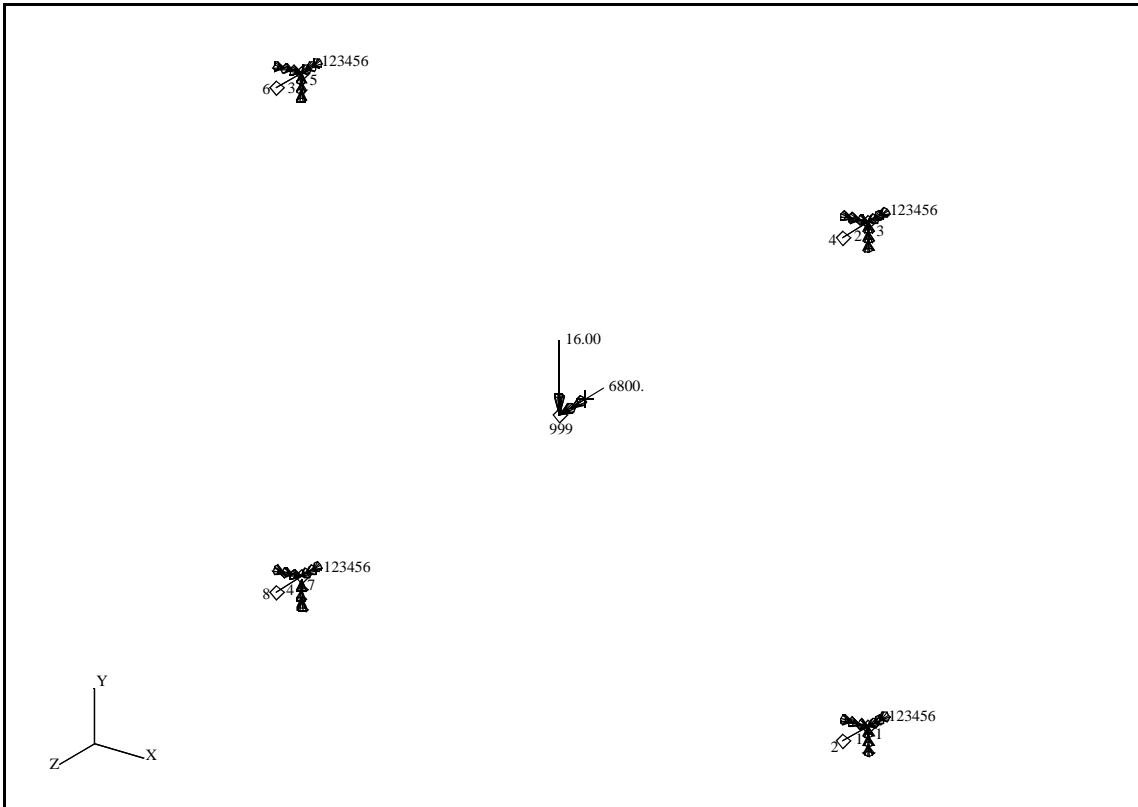
*(See Figure D.4)*

Add

OK

Apply

**Figure 10a.5 - Forces and Boundry Conditions**



12. Create the rigid element.

◆ **Finite Elements**

Action:

Create

Object:

MPC

Type:

RBE3

Define Terms...

◆ **Create Dependent**

Auto Execute

Node List

Node 999

Select DOFs by holding the Shift key down while clicking with the left mouse button. .

*DOFs*

<b>UX</b>
<b>UY</b>
<b>UZ</b>
<b>RX</b>
<b>RY</b>
<b>RZ</b>

<b>Apply</b>
--------------

◆ **Create Independent**

*Node List*  
(See Figure D.4)

<b>Node 2:8:2</b>
-------------------

*DOFs*

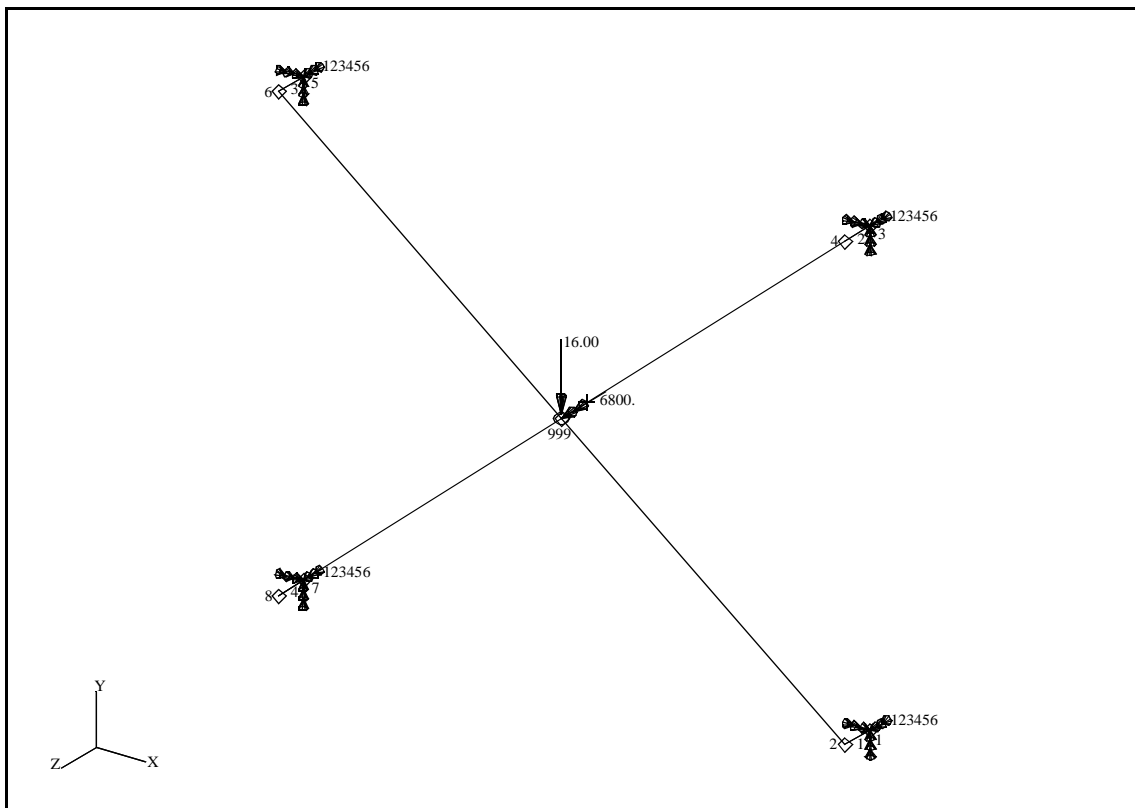
<b>UX</b>
<b>UY</b>
<b>UZ</b>

<b>Apply</b>
--------------

<b>Cancel</b>
---------------

<b>Apply</b>
--------------

**Figure 10a.6 - RBE3 Elements, Loads, and Boundary Conditions**



13. Choose the desired analysis for Case1.

◆ **Load Cases**

Action:

Create

Load Case Name:

Case\_1

**Assign/Prioritize Loads/BCs**

Select Loads/BCs to Add to Spreadsheet:

Displ\_Fixed\_ends  
Force\_Shear\_Reaction

OK

Apply

14. Choose the desired analysis for Case2. .

◆ **Load Cases**

Action:

Create

*Load Case Name:*

**Assign/Prioritize Loads/BCs**

*Select Loads/BCs to Add to Spreadsheet:*

*Load/BC Name:*

**Remove Selected Row**

15. Now you are ready to run the analysis.

◆ **Analysis**

*Action:*

*Object:*

*Method:*

*Jobname*

*Solution Type*

*Available Subcases:*

*Select Result Type*

*Available Subcases:*

◆ **Linear Static**

(Click on each item once to select.)

---

*Select Result Type*

**Multi-Point Constraint Forces**  
**Element Forces**  
**Applied Load**  
**Grid Point Force Balance**

**OK**

**Apply**

**Cancel**

**Subcase Select...**

*Subcases For Solution  
Sequence: 101*

**Case\_1**  
**Case\_2**

(Click on these to select.)

*Subcases Selected:*

**Default**

(Click on this to deselect.)

**OK**

**Apply**

An MSC.Nastran input file called **bolt\_load.bdf** will be generated. This process of translating your model into an input file is called the Forward Translation. The Forward Translation is complete when the Heartbeat turns green. MSC.Patran Users should proceed to Step 16.

## Generating an input file for MSC.Nastran Users:

MSC.Nastran users can generate an input file using the data from table 10a.1 and figure 10a.1. The result should be similar to the output below.

### 16. MSC.Nastran input file: **bolt\_load.bdf**

```

ID SEMINAR, APPENDIX D
SOL 101
TIME 600
CEND
TITLE = Shear and Moment Reactions - Linear Static Analysis with RBE3
SUBCASE 1
  SPC = 2
  LOAD = 2
  DISPLACEMENT ( SORT1 , REAL ) = ALL
  SPCFORCES ( SORT1 , REAL ) = ALL
  OLOAD ( SORT1 , REAL ) = ALL
  MPCFORCES ( SORT1 , REAL ) = ALL
  GPFORCE = ALL
  STRESS ( SORT1 , REAL , VONMISES , BILIN ) = ALL
  FORCE ( SORT1 , REAL , BILIN ) = ALL
SUBCASE 2
  SPC = 2
  LOAD = 4
  DISPLACEMENT ( SORT1 , REAL ) = ALL
  SPCFORCES ( SORT1 , REAL ) = ALL
  OLOAD ( SORT1 , REAL ) = ALL
  MPCFORCES ( SORT1 , REAL ) = ALL
  GPFORCE = ALL
  STRESS ( SORT1 , REAL , VONMISES , BILIN ) = ALL
  FORCE ( SORT1 , REAL , BILIN ) = ALL
BEGIN BULK
PARAM   POST      -1
PARAM   PATVER    3.
PARAM   AUTOSPC   YES
PARAM   INREL     0
PARAM   ALTRED    NO
PARAM   COUPMASS -1
PARAM   K6ROT     0.
PARAM   WTMASS    1.
PARAM, NOCOMPS, -1
PARAM   PRTMAXIM  YES
PBARL   1         1             TUBE                +      A
+      A 2.       1.9
CBAR    1         1           1           2           999
CBAR    2         1           3           4           999
CBAR    3         1           5           6           999
CBAR    4         1           7           8           999
MAT1*   1         7.1+10      .3                *      B
*      B
RBE3    5         999         123456  1.         123         2         4         +      C
+      C 6         8
GRID    1         75.        -60.        0.
GRID    2         75.        -60.        10.
GRID    3         75.         60.         0.
GRID    4         75.         60.        10.

```

---

```

GRID      5          -75.    60.    0.
GRID      6          -75.    60.   10.
GRID      7          -75.   -60.    0.
GRID      8          -75.   -60.   10.
GRID     999           0.     0.   10.
SPCADD    2          3
LOAD      2          1.     1.     1
LOAD      4          1.     1.     3     1.     5
SPC1      3          123456  1     3     5     7
FORCE     1          999     0     16.    0.    -1.    0.
FORCE     3          999     0     0.     .57735 .57735 .57735
MOMENT    5          999     0     6800.  0.     0.     1.
ENDDATA

```

## Submit the input file for analysis:

17. Submit the input file to MSC.Nastran for analysis.
  - 17a. To submit the MSC.Patran **.bdf** file for analysis, find an available UNIX shell window. At the command prompt enter: **nastran bolt\_load.bdf scr=yes**. Monitor the run using the UNIX **ps** command.
  - 17b. To submit the MSC.Nastran **.dat** file for analysis, find an available UNIX shell window. At the command prompt enter: **nastran bolt\_load scr=yes**. Monitor the run using the UNIX **ps** command.
18. When the run is completed, edit the **bolt\_load.f06** file and search for the word **FATAL**. If no matches exist, search for the word **WARNING**. Determine whether existing **WARNING** messages indicate modeling errors.
19. While still editing **bolt\_load.f06**, search for the word:  
**M U L T** (spaces are necessary)

## Comparison of Results:

20. Compare the results obtained in the .f06 file with the results on the following page:



**21. Result Verification**

Subcase 1 (Point 2)

$$T_2 = -4\text{kN}$$

$$(F' = 4 \text{ kN, Pg.5})$$

Subcase 2 (Point 2)

$$T_1 = 11.057\text{kN}$$

$$T_2 = 13.82\text{kN}$$

$$F'' = \sqrt{(T_1)^2 + (T_2)^2} = \sqrt{(11.057)^2 + (13.82)^2}$$

$$F'' = 17.70\text{kN}$$

$$(F'' = 17.7\text{kN, Pg. 5})$$

---

22. MSC.Nastran Users have finished this exercise. MSC.Patran Users should proceed to the next step.

23. Proceed with the Reverse Translation process, that is importing the **bolt\_load.op2** results file into MSC.Patran. To do this, return to the *Analysis* form and proceed as follows:

◆ **Analysis**

<i>Action:</i>	<b>Read Output2</b>
<i>Object:</i>	<b>Result Entities</b>
<i>Method</i>	<b>Translate</b>
<b>Select Results File...</b>	
<i>Select Results File</i>	<b>bolt_load.op2</b>
<b>OK</b>	
<b>Apply</b>	

24. When the translation is complete bring up the **Results** form.

Select **Fringe** to view different results with color spectrum analysis.

◆ **Results**

<i>Action:</i>	<b>Create</b>
<i>Object:</i>	<b>Fringe</b>

To select results, click on the **Select Results** icon.



**Select Result**

<i>Select Result Case(s):</i>	<b>Case_1, Static Subcase</b>
<i>Select Fringe Result:</i>	<b>Grid Point Forces, Total</b>

To change the display attributes of the plot, click on the **Display Attributes** icon.



**Display Attributes**

*Style:*   
*Element Shrink Factor*   
*Display:*   
*Style:*   
  
*Label Format*   
*Significant figures*

Select **Deformation** to view physical changes of the model.

◆ **Results**

*Action:*   
*Object:*

To select results, click on the **Select Results** icon.



*Select Result Case(s):*   
*Select Deformation Result:*

To change the display attributes of the plot, click on the **Display Attributes** icon.



- Show Undeformed**
- Show Max/Min Label**

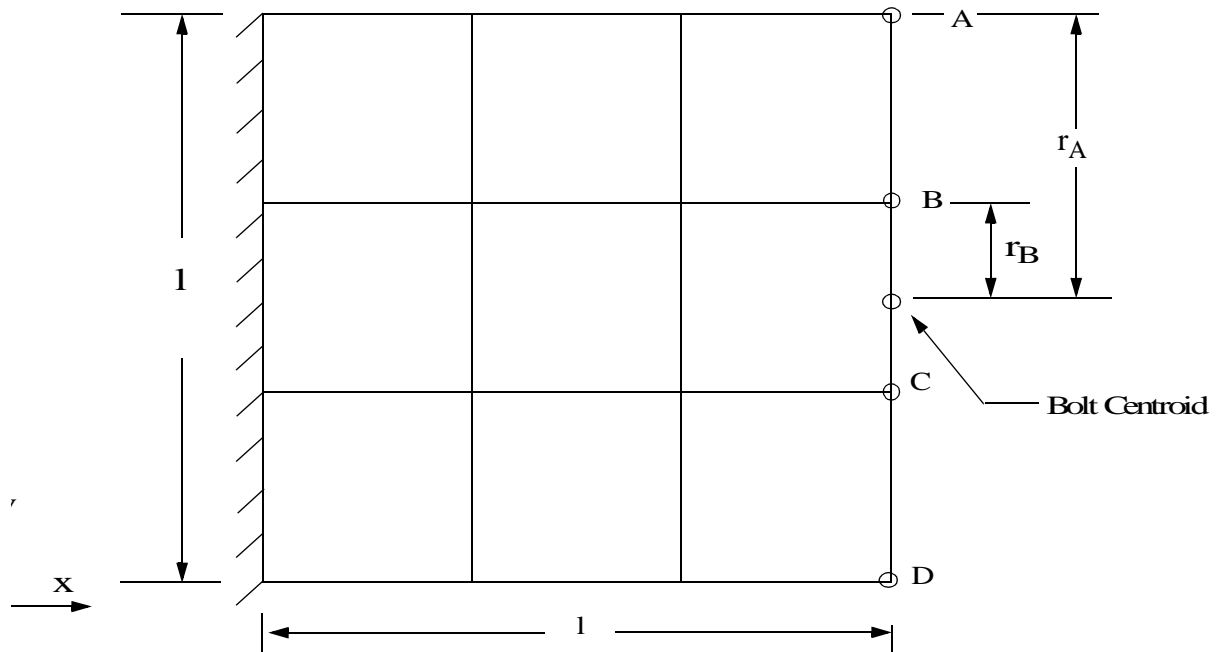
- 
25. If you wish to reset your display graphics to the state it was in before you began post-processing your model, remember to select the **Reset Graphics** icon.



**Reset Graphics**

To view different results, after **Reset Graphics** repeat step 23 and change *Result Case(s)*, *Fringe Result*, and *Deformation Result*.

Quit MSC.Patran when you are finished with this exercise.



$$M = F_A''r_A + F_B''r_B + F_C''r_C + F_D''r_D$$

Since  $\frac{F_A''}{r_A} = \frac{F_B''}{r_B} = \frac{F_C''}{r_C} = \frac{F_D''}{r_D}$

$$F_n'' = \frac{Mr_n}{r_A^2 + r_B^2 + r_C^2 + r_D^2}$$

Reference: Shigley & Mischke, Mechanical Engineering Design, Fifth Edition  
McGraw-Hill Book Company

---

Sample Calculation:

$$\begin{aligned}F_A'' &= (1000 \cdot 0.5) / (0.5^2 + 0.1667^2 + 0.1667^2 + 0.5^2) \\ &= (1000 \cdot 0.5) / (0.555) \\ &= 900\end{aligned}$$

$$\begin{aligned}F_B'' &= \frac{(1000 \cdot 0.1667)}{(0.5^2 + 0.1667^2 + 0.1667^2 + 0.5^2)} \\ &= 300\end{aligned}$$

```

SOL 101
CEND
ECHO = NONE
SUBCASE 1
    SPC                1
    LOAD              1
    MPCFORCE=all
SUBCASE 2
    SPC = 1
    LOAD = 2
    MPCFORCE=all
SUBCASE 3
    SPC = 1
    LOAD = 3
    MPCFORCE=all
SUBCASE 4
    SPC = 1
    LOAD = 4
    MPCFORCE=all
SUBCASE 5
    SPC                1
    LOAD              5
    MPCFORCE=all
SUBCASE 6
    SPC                1
    LOAD              6
    MPCFORCE=all
BEGIN BULK

PARAMPOST -
PSHELL    1    1    .1    1    1

CQUAD4    1    1    1    2    6    5
CQUAD4    2    1    2    3    7    6
CQUAD4    3    1    3    4    8    7
CQUAD4    4    1    5    6    10   9
CQUAD4    5    1    6    7    11   10
CQUAD4    6    1    7    8    12   11
CQUAD4    7    1    9    10   14   13
CQUAD4    8    1    10   11   15   14
CQUAD4    9    1    11   12   16   15

MAT1      1    1.+7    .3
RBE3      10   999    123456  1.    12354    8
+
A
+
A 12    16

GRID      1    0.    0.    0.
GRID      2    .333333  0.    0.
GRID      3    .666667  0.    0.
GRID      4    1.    0.    0.
GRID      5    0.    .333333  0.
GRID      6    .333333  .333333  0.
GRID      7    .666667  .333333  0.
GRID      8    1.    .333333  0.
GRID      9    0.    .666667  0.
GRID     10    .333333  .666667  0.
    
```

---

```

GRID      11      .666667      .666667      0.
GRID      12       1.        .666667      0.
GRID       3       0.         1.          0.
GRID       4      .333333      1.          0.
GRID       5      .666667      1.          0.
GRID      16       1.         1.          0.
GRID     999       1.         .5          0.

spCi      1      123456  1         5          9          13

FORCE     1       999      0         1000.      1.         0.         0.
FORCE     2       999      0         1000.      0.         1.         0.
FORCE     3       999      0         1000.      0.         0.         1.
MOMENT    4       999      0         1000.      1.         0.         0.
MOMENT    5       999      0         1000.      0.         1.         0.
MOMENT    6       999      0         1000.      0.         0.         1.

ENDDATA

```

Question: What happen if RBE3 is modified to reflect the following independent DOFs?

```

0                                SUBCASE 1
      F O R C E S O F      M U L T I P O I N T      C O N S T R A I N T

POINT ID.      TYPE      T1      T2      T3      R1      R2      R3
   4           G      2.500000E+02  0.0  0.0      0.0      0.0  0.0
   8           G      2.500000E+02  0.0  0.0      0.0      0.0  0.0
  12           G      2.500000E+02  0.0  0.0      0.0      0.0  0.0
  16           G      2.500000E+02  0.0  0.0      0.0      0.0  0.0
 999           G      -1.000000E+03  0.0  0.0      0.0      0.0  0.0

0                                SUBCASE 2
      F O R C E S O F      M U L T I P O I N T      C O N S T R A I N T

POINT ID.      TYPE      T1      T2      T3      R1      R2      R3
   4           G      0.0      2.500000E+02  0.0  0.0      0.0  0.0
   8           G      0.0      2.500000E+02  0.0  0.0      0.0  0.0
  12           G      0.0      2.500000E+02  0.0  0.0      0.0  0.0
  16           G      0.0      2.500000E+02  0.0  0.0      0.0  0.0
 999           G      0.0      -1.000000E+03  0.0  0.0      0.0  0.0

0                                SUBCASE 3
      F O R C E S O F      M U L T I P O I N T      C O N S T R A I N T

POINT ID.      TYPE      T1      T2      T3      R1      R2      R3
   4           G      0.0      0.0      2.500000E+02  0.0  0.0  0.0
   8           G      0.0      0.0      2.500000E+02  0.0  0.0  0.0
  12           G      0.0      0.0      2.500000E+02  0.0  0.0  0.0
  16           G      0.0      0.0      2.500000E+02  0.0  0.0  0.0
 999           G      0.0      0.0      -1.000000E+03  0.0  0.0  0.0

```

```

0
      SUBCASE 4
      F O R C E S      O F      M U L T I P O I N T C O N S T R A I N T

POINT ID.      TYPE      T1      T2      T3      R1      R2      R3
   4           G      0.0    0.0   -8.999996E+02   0.0      0.0     0.0
   8           G      0.0    0.0   -3.000005E+02   0.0      0.0     0.0
  12           G      0.0    0.0    3.000005E+02   0.0      0.0     0.0
  16           G      0.0    0.0    8.999996E+02   0.0      0.0     0.0
  999          G      0.0    0.0      0.0   -1.000000E+03   0.0     0.0

0
      SUBCASE 5
      F O R C E S      O F      M U L T I P O I N T C O N S T R A I N T

POINT ID.      TYPE      T1      T2      T3      R1      R2      R3
   4           G      0.0    0.0    0.0      0.0      2.500000E+02   0.0
   8           G      0.0    0.0    0.0      0.0      2.500000E+02   0.0
  12           G      0.0    0.0    0.0      0.0      2.500000E+02   0.0
  16           G      0.0    0.0    0.0      0.0      2.500000E+02   0.0
  999          G      0.0    0.0    0.0      0.0     -1.000000E+03   0.0

0
      SUBCASE 6
      F O R C E S      O F      M U L T I P O I N T C O N S T R A I N T

POINT ID.      TYPE      T1      T2      T3      R1      R2      R3
   4           G    8.999996E+02   0.0  0.0      0.0    0.0      0.0
   8           G    3.000005E+02   0.0  0.0      0.0    0.0      0.0
  12           G   -3.000005E+02   0.0  0.0      0.0    0.0      0.0
  16           G   -8.999996E+02   0.0  0.0      0.0    0.0      0.0
  999          G      0.0      0.0  0.0      0.0    0.0   -1.00000000E+03

```